

第三届LS-DYNA中国论坛

2018年10月26日 上海



Livermore Software Technology Corporation



上海仿坤软件科技有限公司 (LS-DYNA 中国)



Application of LS-DYNA in safety product developing

Autoliv CTC in Sep 19th, 2018
v 1.0

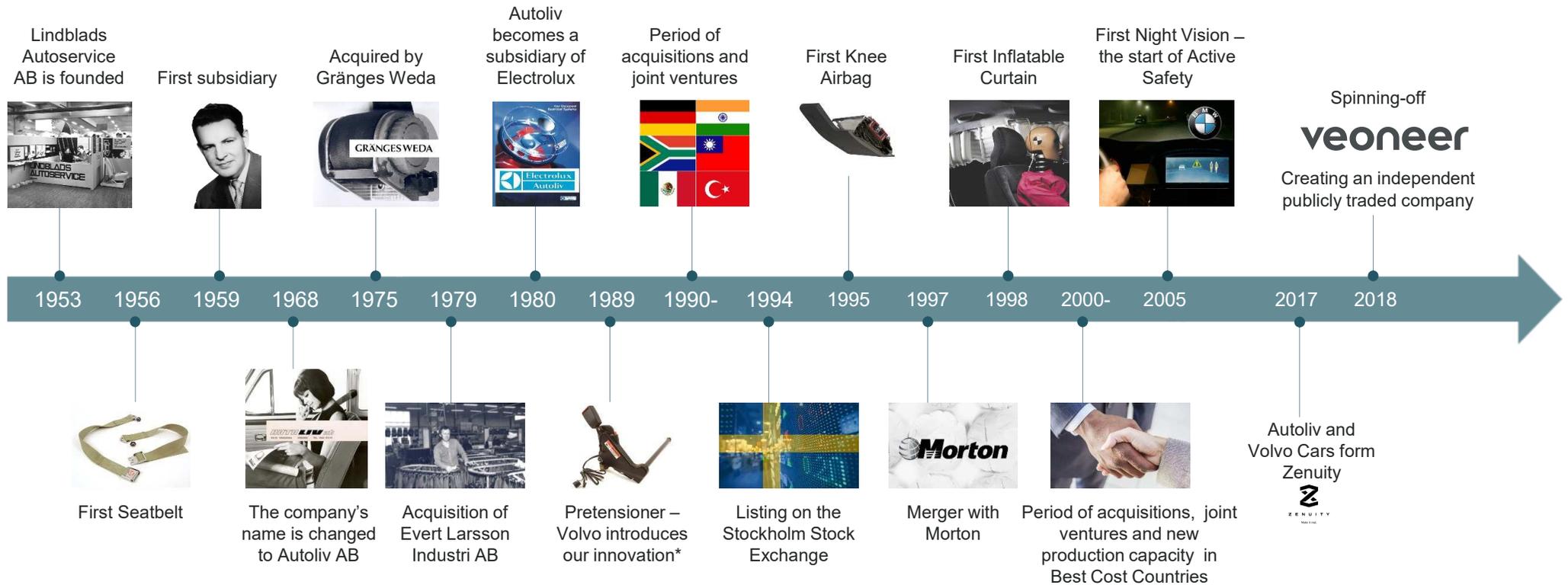


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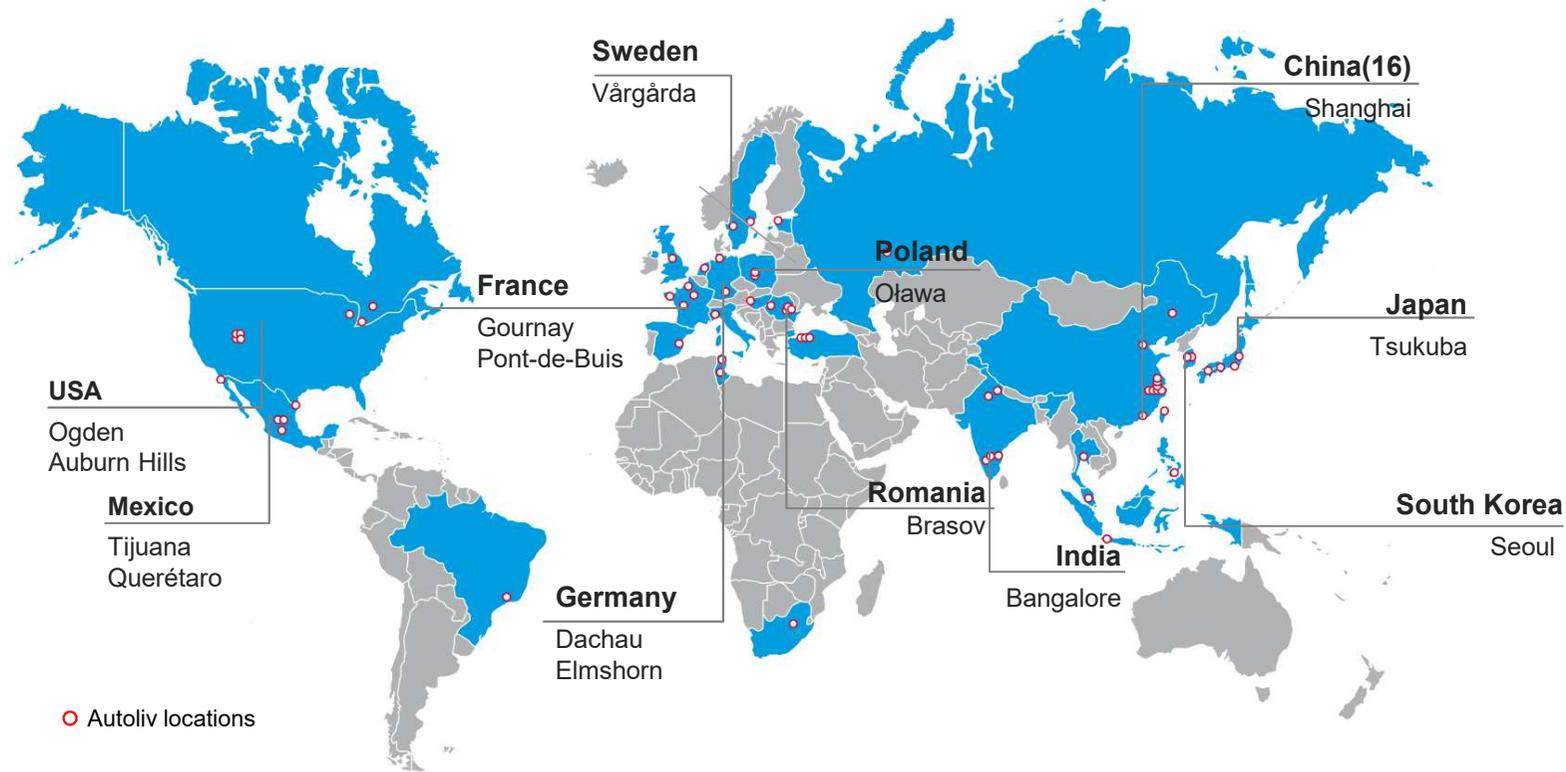
Contents

- Autoliv
- Safety product
- Strength
- Airbag deployment
- Fatigue
- Acoustic

Autoliv – 65 years of Dedication to Saving Lives



Autoliv's Research, Development & Engineering



27 Countries

Across All Key Automotive Regions

64 Facilities

In Close Proximity to OEMs

12 Tech Centers

With 5,300 Engineers, Close to OEMs' Engineering Hubs

19 Crash Test Tracks

Enabling Crash Simulations Globally

Safety Technologies



Airbags

In frontal crashes, driver airbags reduce fatalities by appr 25%, and front-seat airbags by appr 20%. Side-curtain airbags reduce life-threatening head injuries in side impacts by appr 50% for passengers sitting on the side of the vehicle that is struck. Side airbags, rear side airbags, knee airbags and far-side airbags also help save lives and reduce serious injuries.

Seatbelts prevent fatalities

The combination of seatbelt, seatbelt pretensioners, load limiters, lap pretensioners and frontal airbags reduces the risk of life-threatening head or chest injuries in frontal crashes by 75%.

Protecting Pedestrians

Thanks to a pyrotechnic hood-lifter and/or an outside pedestrian protection airbag, a pedestrian who is hit can avoid head injury caused by the hard area between the hood and the windshield or one of the windshield pillars.

Safety Product

Airbags – Snapshot

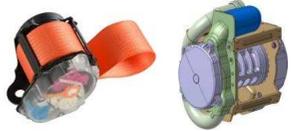
Product Portfolio Overview

Airbags		<ul style="list-style-type: none"> ✓ Driver and passenger ✓ Knee ✓ Side and curtain 	<ul style="list-style-type: none"> ✓ Far-Side ✓ Pedestrian protection
Inflators		<ul style="list-style-type: none"> ✓ Pyrotechnic ✓ Stored gas ✓ Hybrid 	
Initiators and Micro Gas Generators		<ul style="list-style-type: none"> ✓ Initiators ✓ Micro gas generators 	
Textiles		<ul style="list-style-type: none"> ✓ Weaving of one-piece cushions ✓ Cut & sew operations 	

Safety Product

Seatbelts – Snapshot

Product Portfolio Overview

<p>Retractors/ Pretensioners</p>		<ul style="list-style-type: none"> ✓ Tightens seatbelt in case of accident ✓ Stops spool that retains seatbelt webbing from rotating
<p>Pre-Pretensioners</p>		<ul style="list-style-type: none"> ✓ Tightens seatbelt prior to crash or during aggressive driving manoeuvres ✓ Soft tightening as warning for ADAS functions
<p>Buckles</p>		<ul style="list-style-type: none"> ✓ Can provide tightening of seatbelt during crash ✓ Illuminated and buckle presenters
<p>Height Adjusters, Pillar Loops & Tongues, Webbing</p>		<ul style="list-style-type: none"> ✓ Accommodate diverse vehicle interior layouts and safety needs

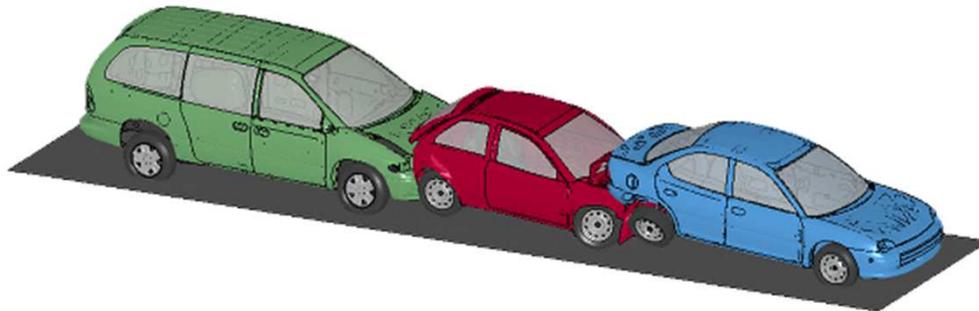
Safety Product

Steering Wheels – Snapshot

Product Portfolio Overview

Steering Wheel Products		<ul style="list-style-type: none">✓ Leather wrap and polyurethane wheels✓ Magnesium & aluminum die cast structures
Steering Wheel Electrical Device		<ul style="list-style-type: none">✓ Heated rim, horn and touch switches✓ Gesture controls✓ Steering wheel ECUs (HOD, heat and alert)
Human Machine Interface (HMI) Features		<ul style="list-style-type: none">✓ HMI platform, enabling ADAS features✓ Optical sensor✓ Hands on/ off detection (HOD)✓ Driver alert (optical and haptic)

Solve all problems in LS-Dyna



One code strategy

- In automotive, one model for crash, durability, NVH shared and maintained across analysis groups.
- Manufacturing simulation results from LS-DYNA used in crash, durability, and NVH modeling.

“All-in-one” package

Structure

Crashworthiness

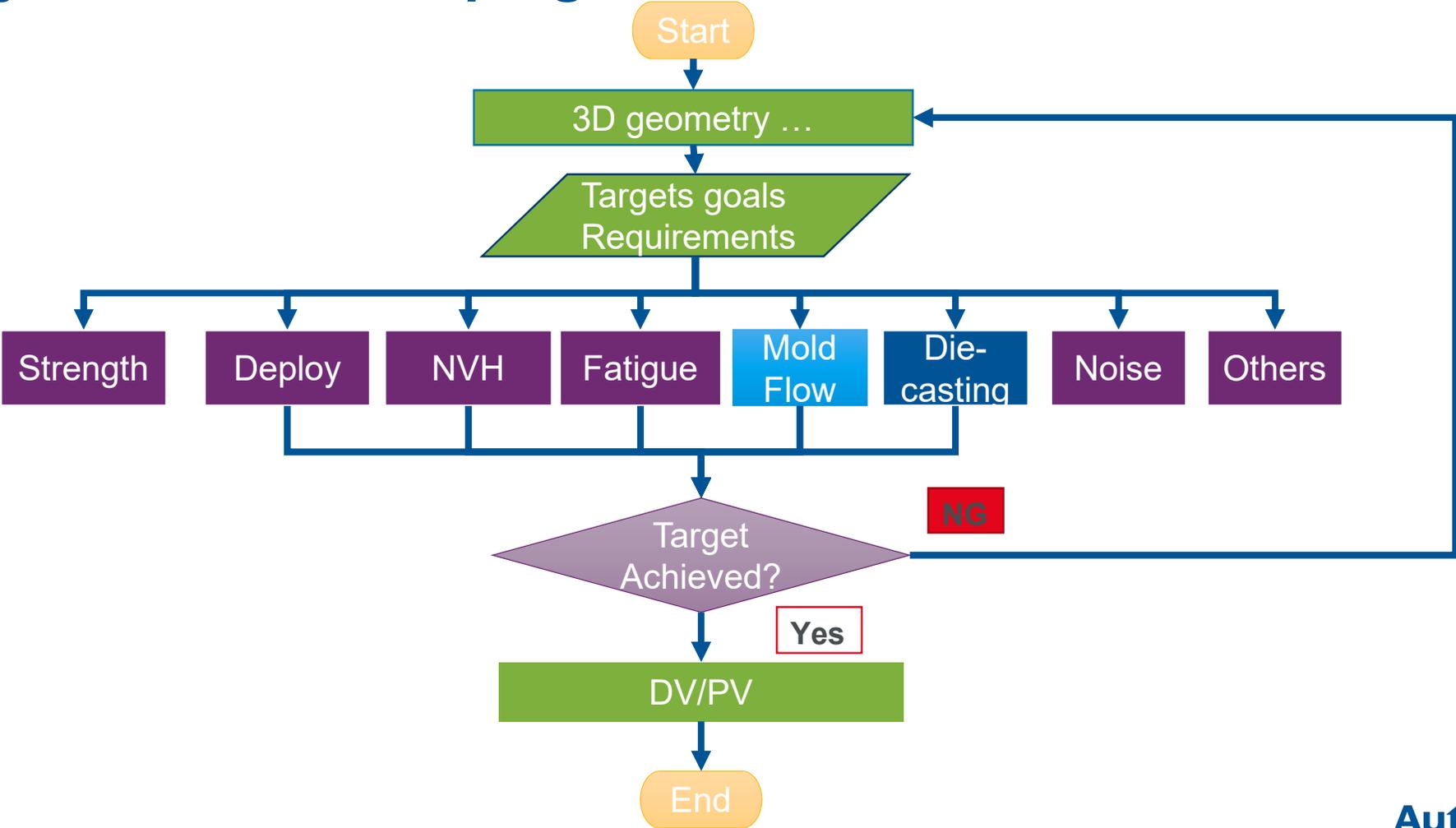
Occupant Safety

NVH

Durability

Acoustic

Safety Product Developing

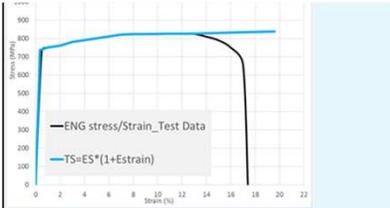


Application of LS-DYAN

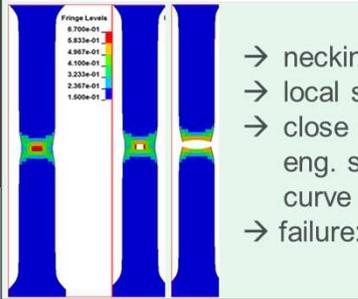
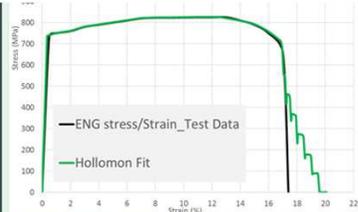
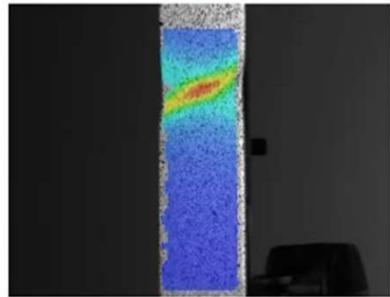
Strength in Implicit and explicit of Ls-Dyna

- Material property
- Seatbelt
- Airbag
- Steering wheel

Strength Analysis -Material property



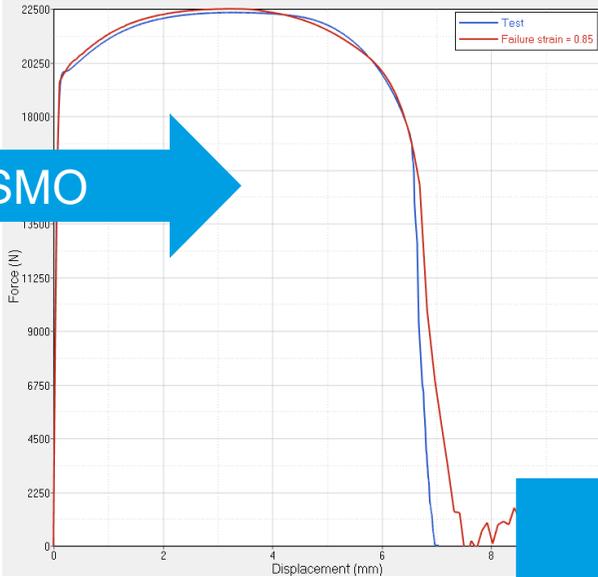
- Uniform Elongation
- No Necking
- No Force Reduction



ARAVIS TEST
→ 68% failure strain

- necking achieved
- local strain (68%)
- close match with eng. stress-strain curve
- failure: inside-out

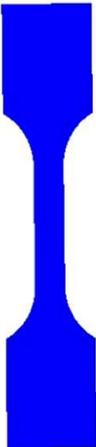
GISSMO



Contour Plot
Strain(vonMises)
Global System
Advanced Average

- 7.551E-01
- 6.712E-01
- 5.873E-01
- 5.034E-01
- 4.195E-01
- 3.366E-01
- 2.517E-01
- 1.678E-01
- 8.390E-02
- 0.000E+00

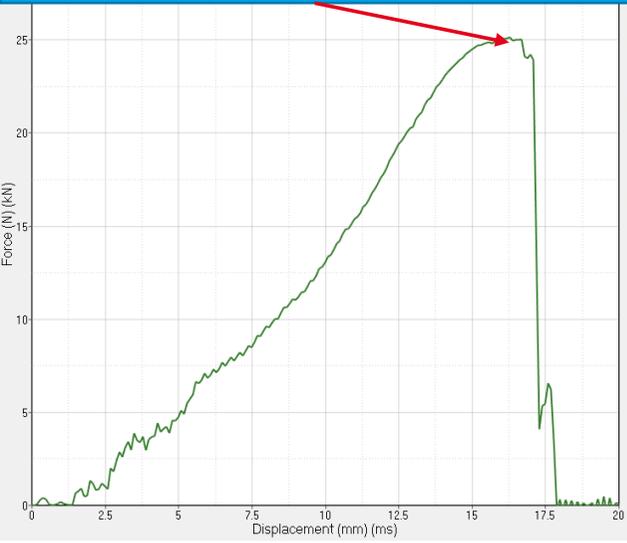
No result
Max = 7.551E-01
Node 6617
Min = 0.000E+00
Node 1304



Validated GISSMO Material Card for strength analysis

Strength Analysis -Seatbelt

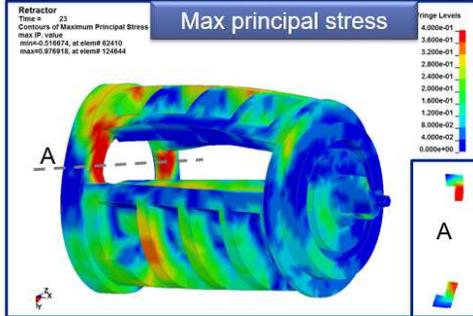
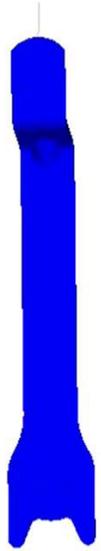
25kN is close to test (24.95 ~26.06)



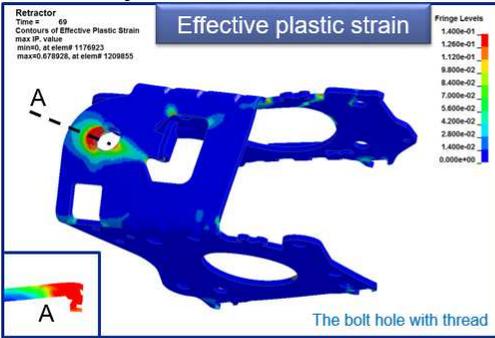
Contour Plot
Strain(vonMises, Max)
Global System
Advanced Average

- 6.376E-01
- 5.667E-01
- 4.959E-01
- 4.250E-01
- 3.542E-01
- 2.834E-01
- 2.125E-01
- 1.417E-01
- 7.084E-02
- 0.000E+00
- No result

Max = 6.376E-01
Node 24288
Min = 0.000E+00
Node 148180



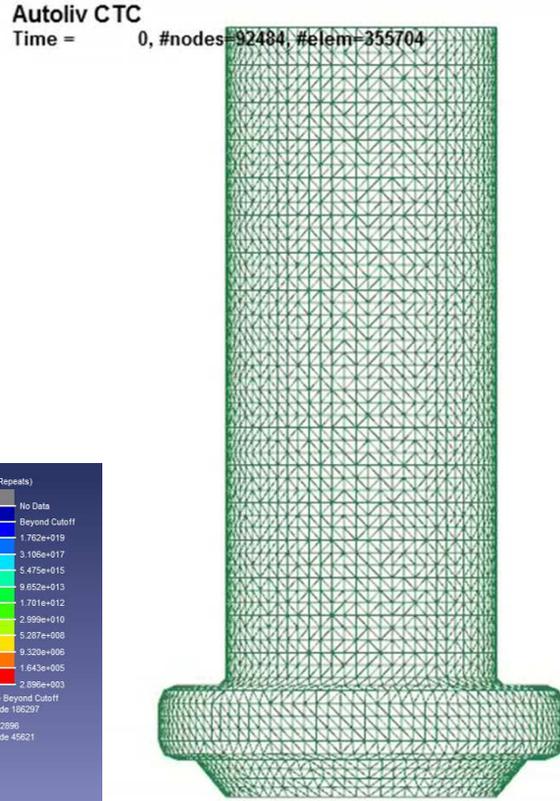
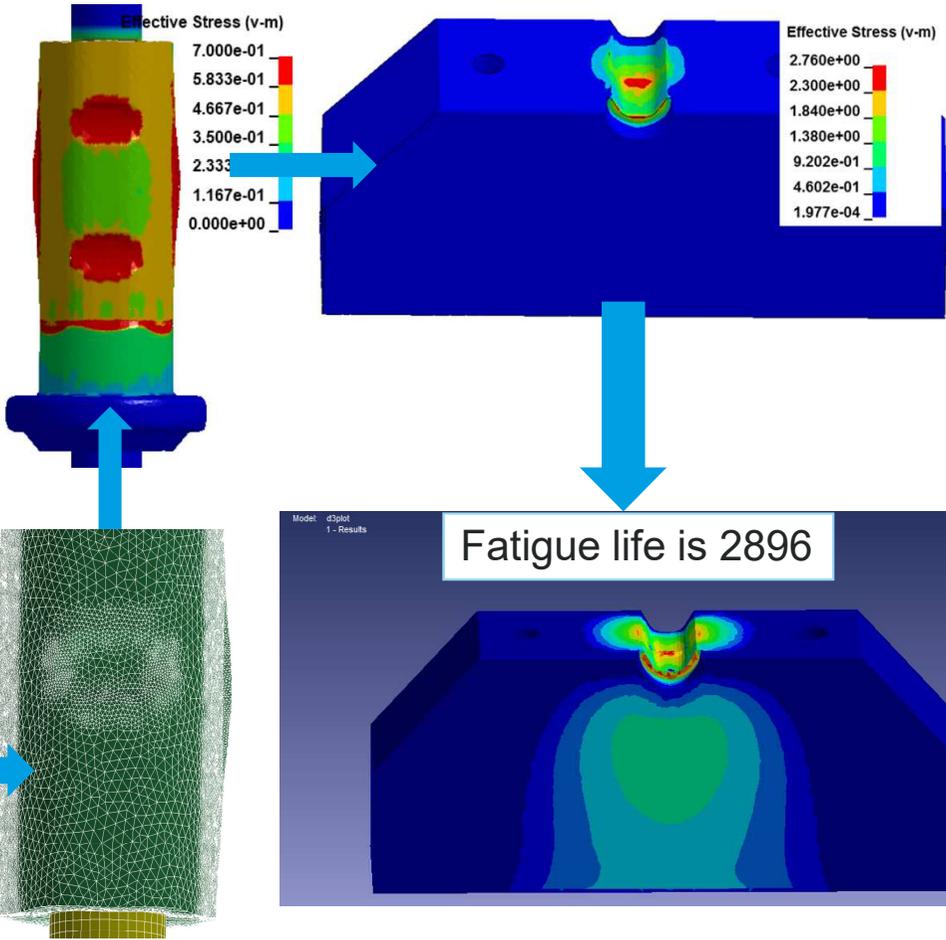
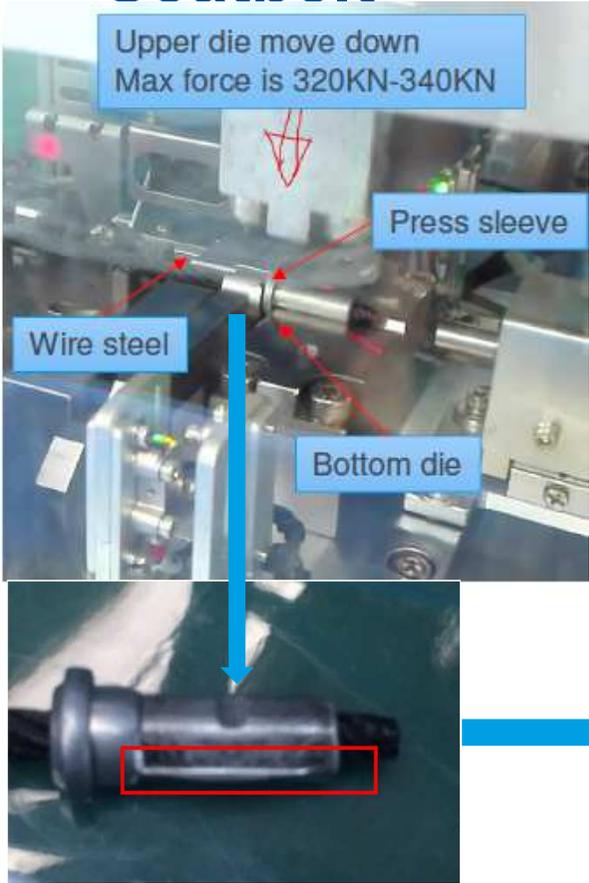
The frame took place fracture when pulling force reached 20kN as similar test result of 17-19kN.



The frame took place fracture when pulling force reached 16kN as similar test result of 16-18kN.

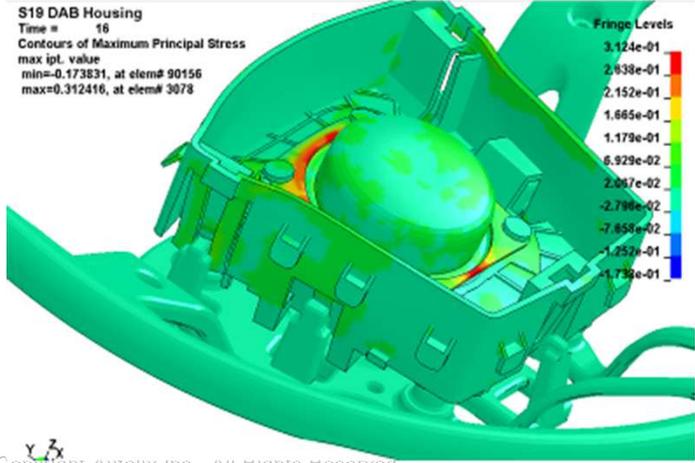
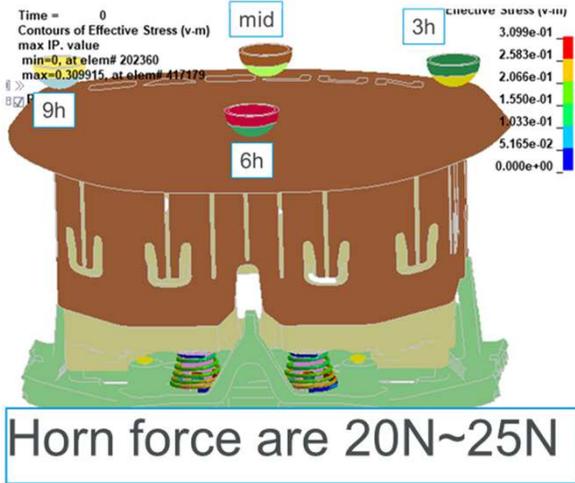
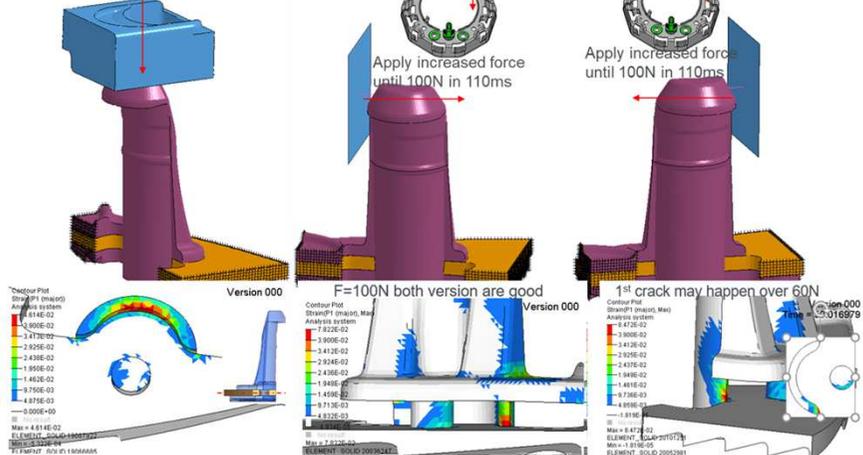
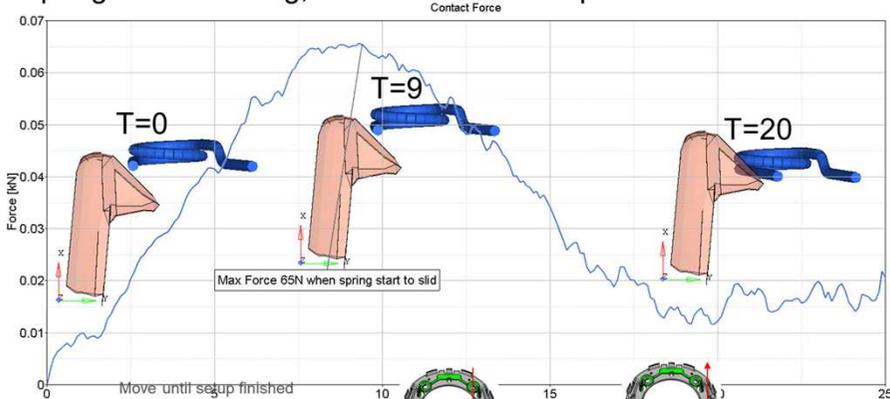
Strength Analysis -Seatbelt

EFG method

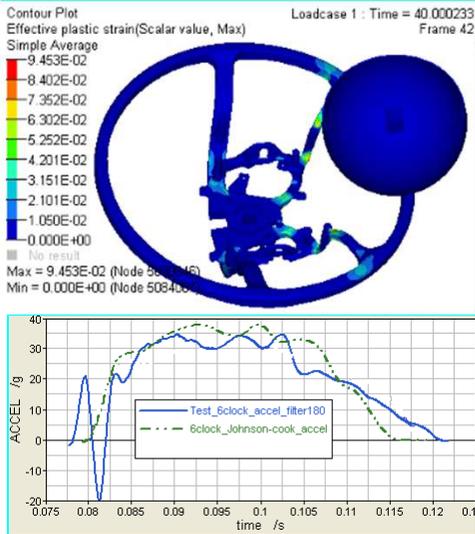
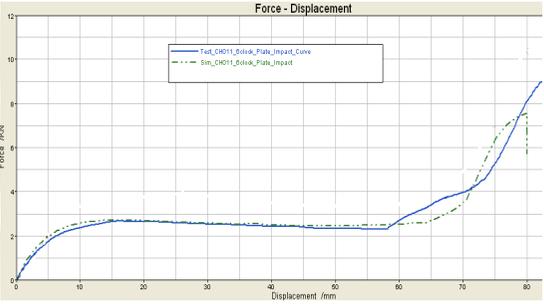
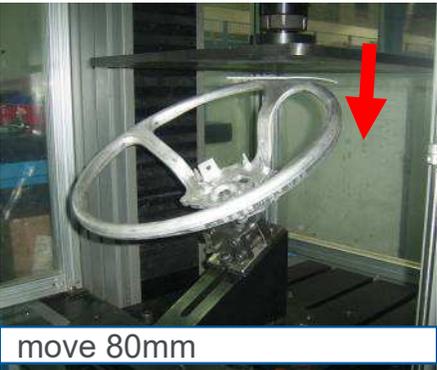


Strength Analysis -Airbag

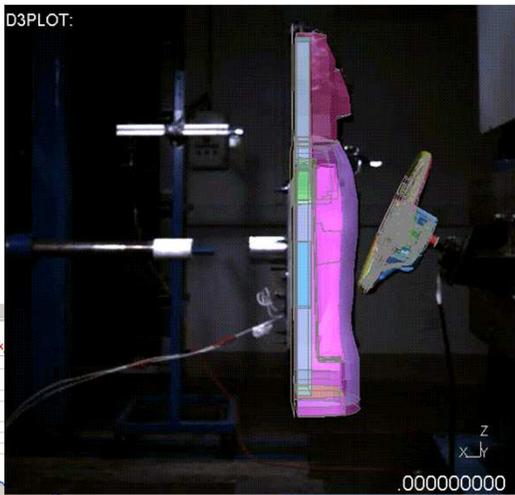
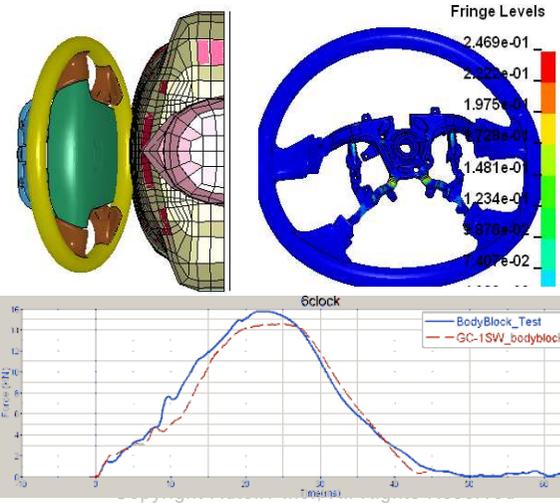
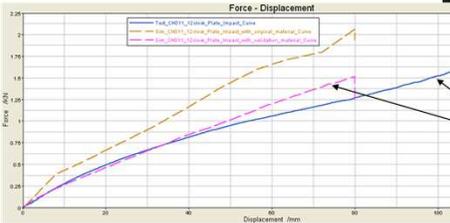
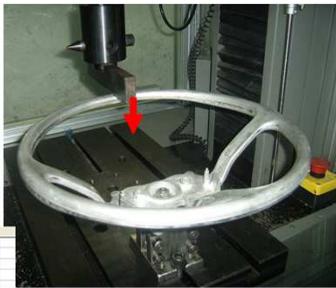
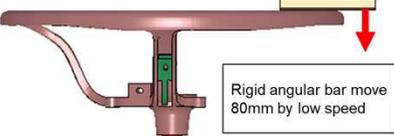
DAB maximum setup force is about 65N when omega spring start to sliding, same with theoretic process.



Strength Analysis -SW



Pressing Stiffness 0° / 12' clock

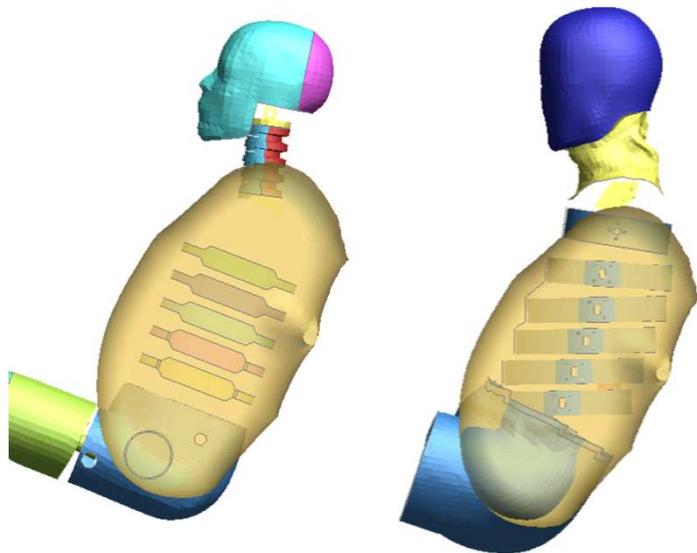


Application of LS-DYAN

Airbag deployment in Ls-Dyna

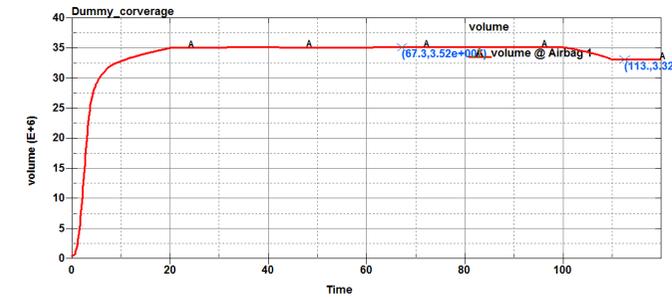
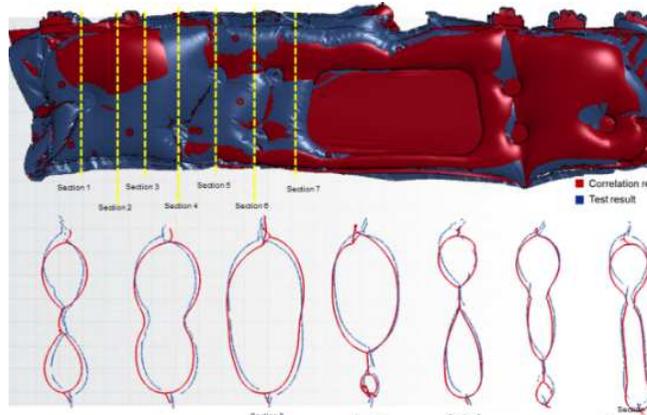
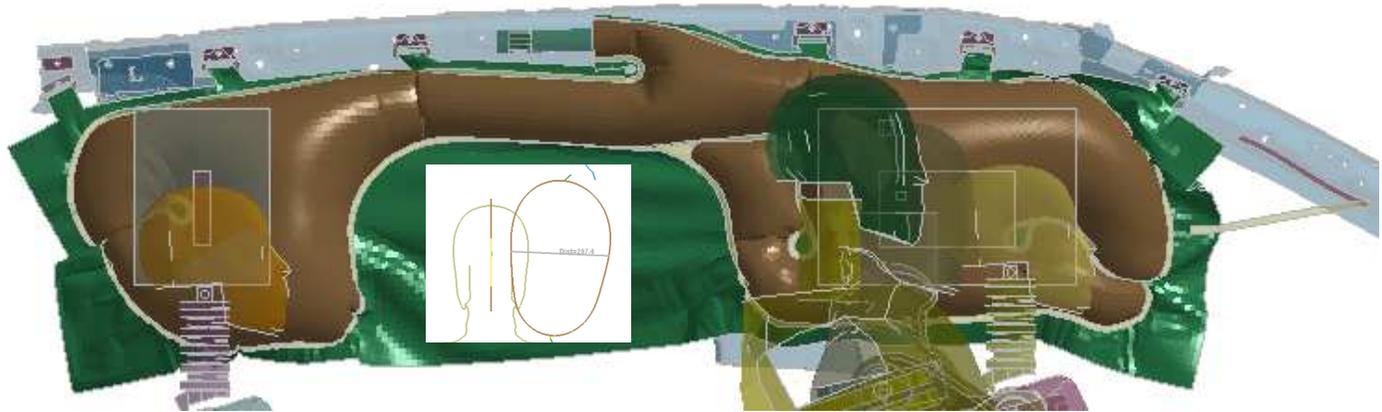
- Coverage area
- Bag deployment
- Airbag subsystem model

Airbag deployment Coverage area

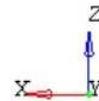
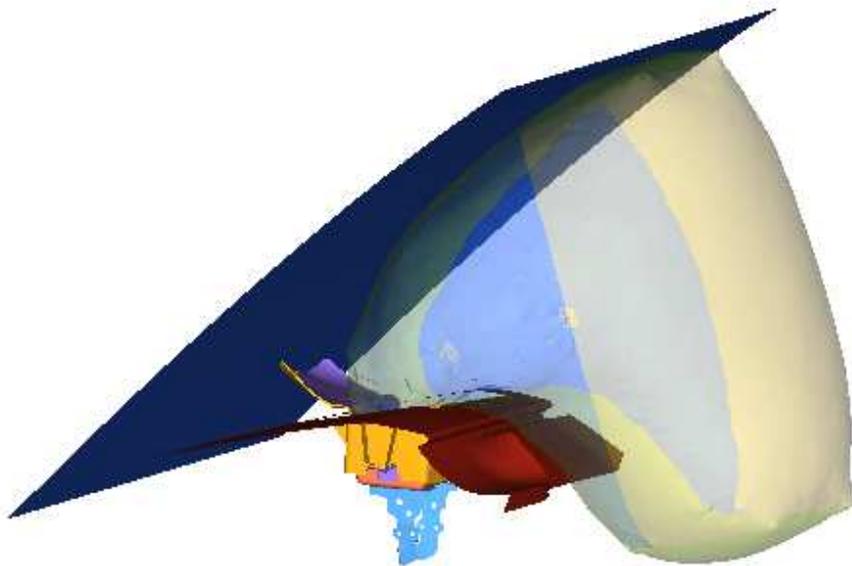


Airbag volume

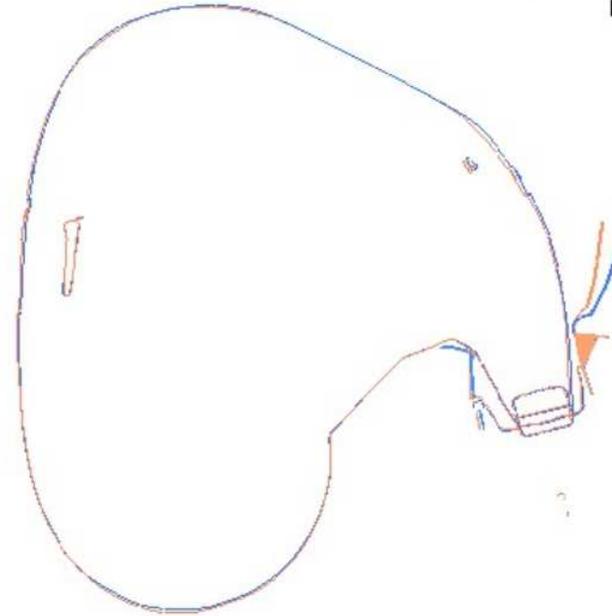
15.3L



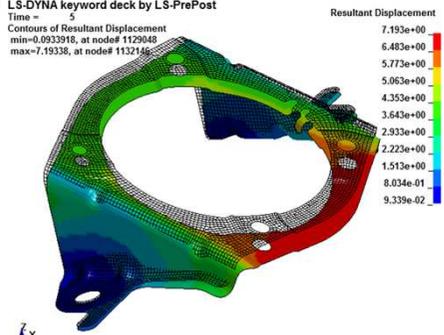
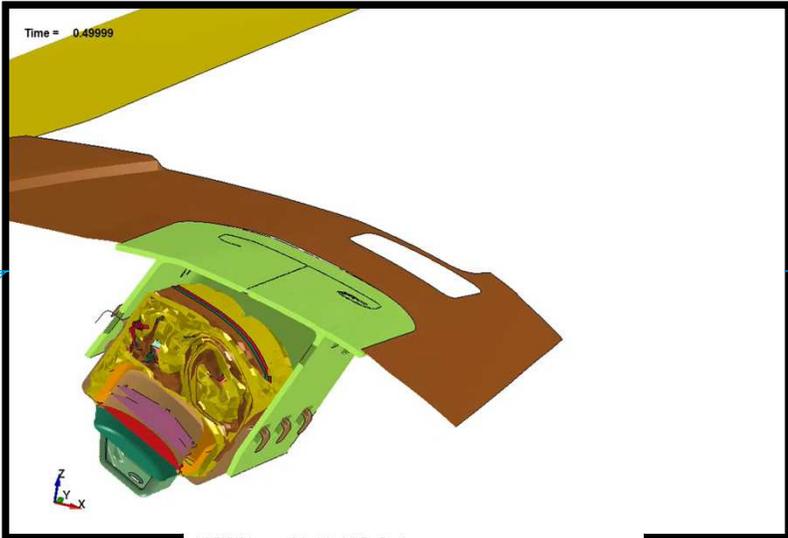
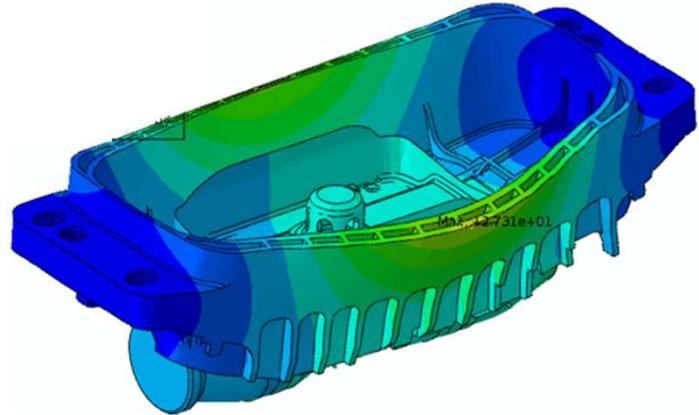
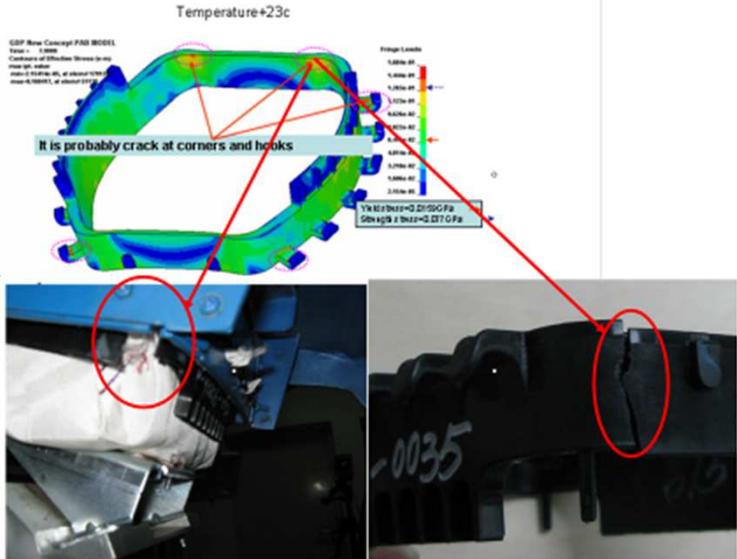
Airbag deployment Coverage area



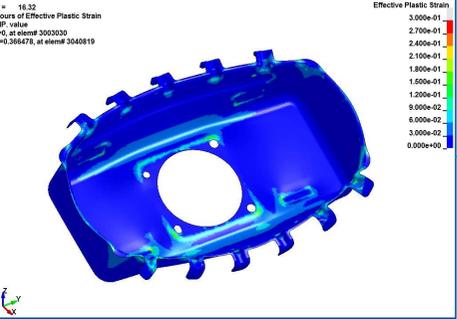
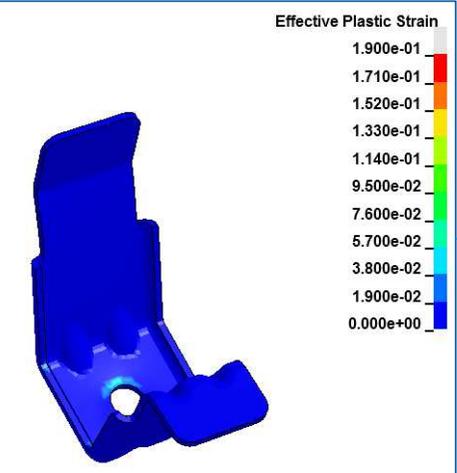
Loadcase 1 : Time = 42.999962
Frame 44



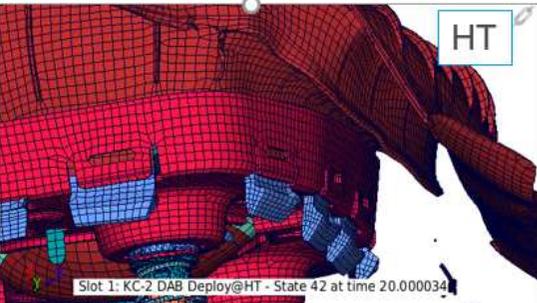
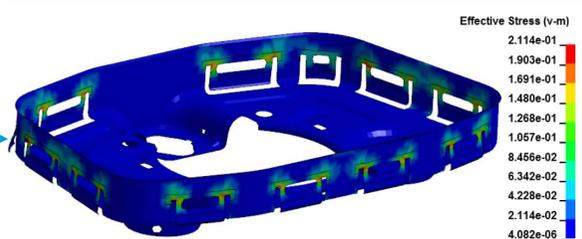
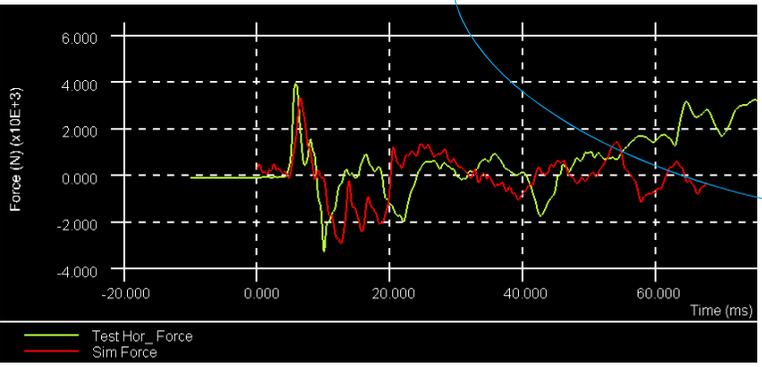
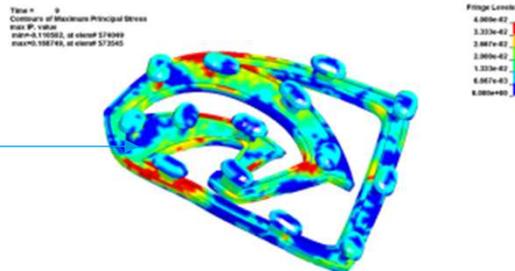
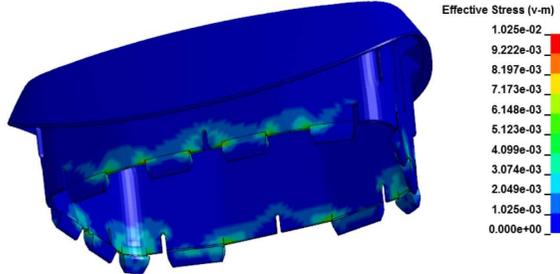
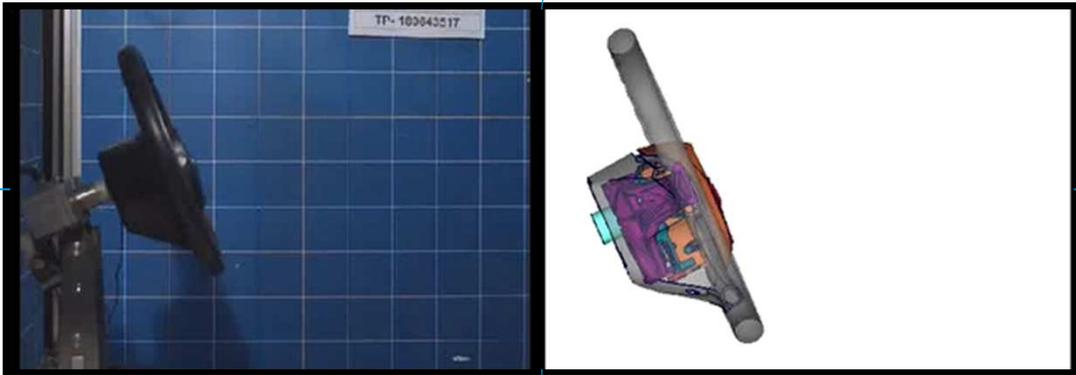
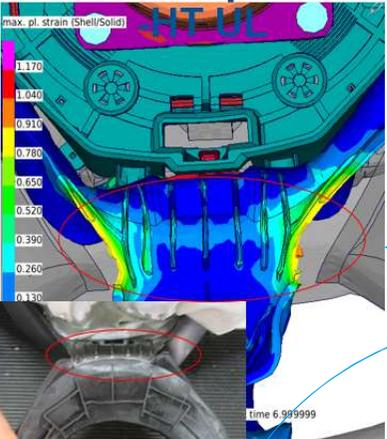
Airbag deployment PAB



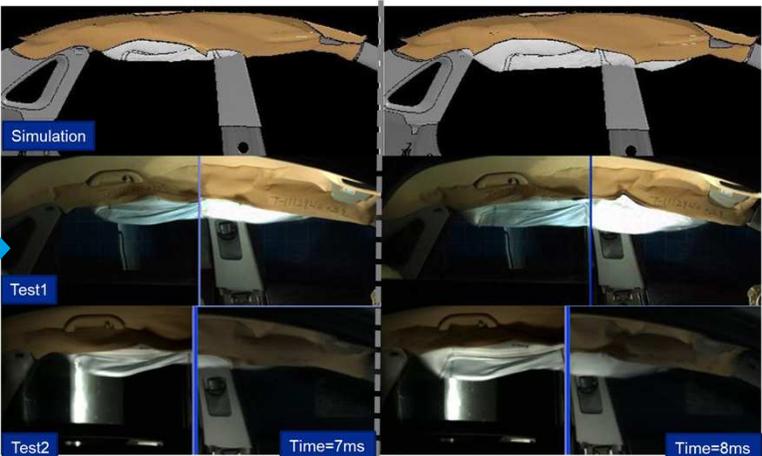
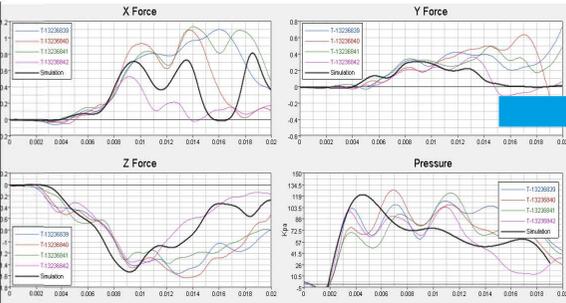
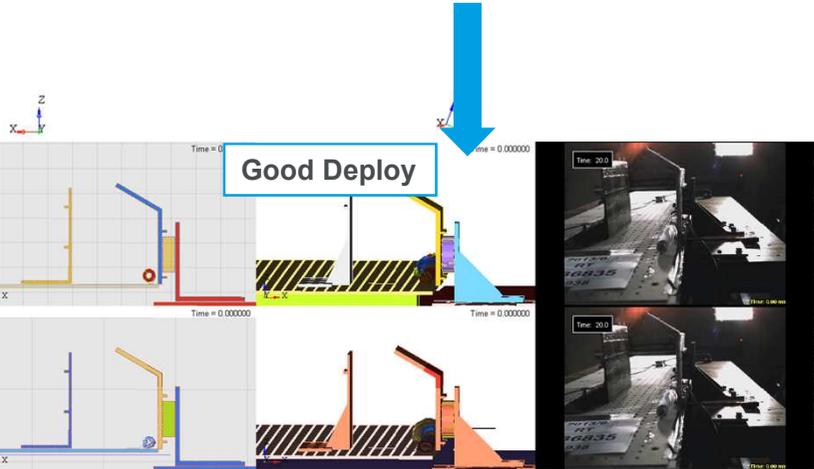
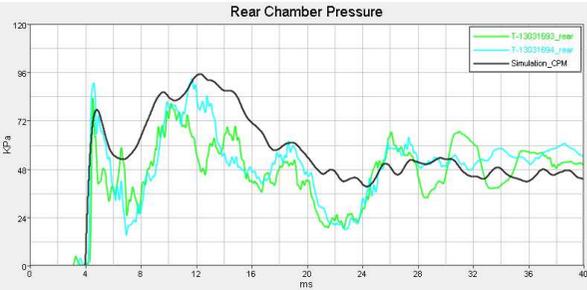
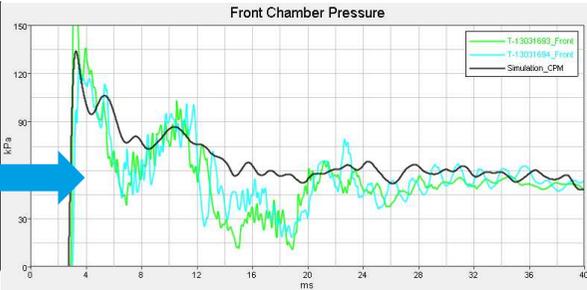
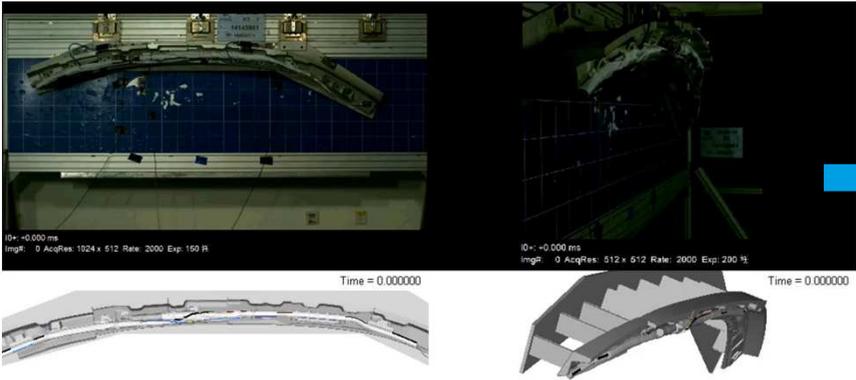
LS-DYNA keyword deck by LS-PrePost
Time = 3
Contours of Resultant Displacement
min=0.0933918, at node# 1129048
max=7.19338, at node# 1132146



Airbag deployment DAB

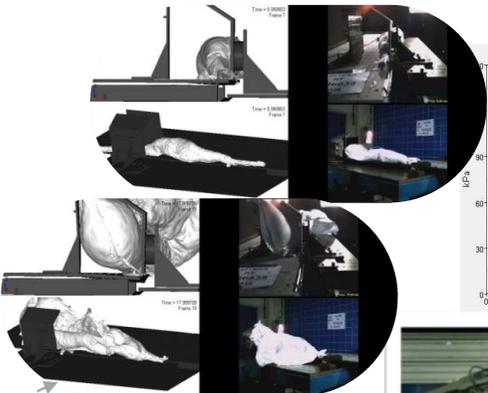
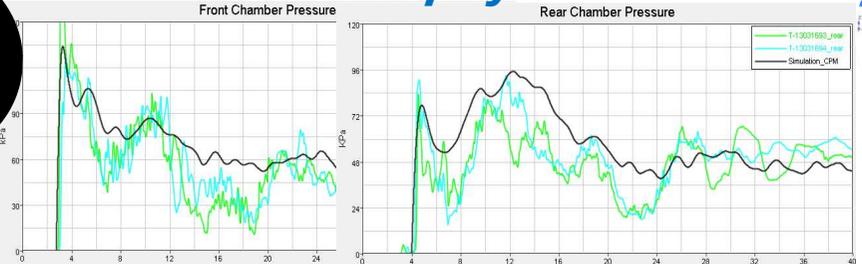


Airbag deployment IC



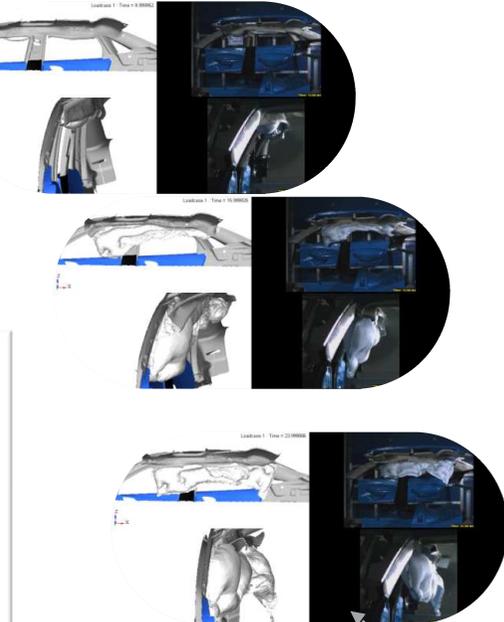
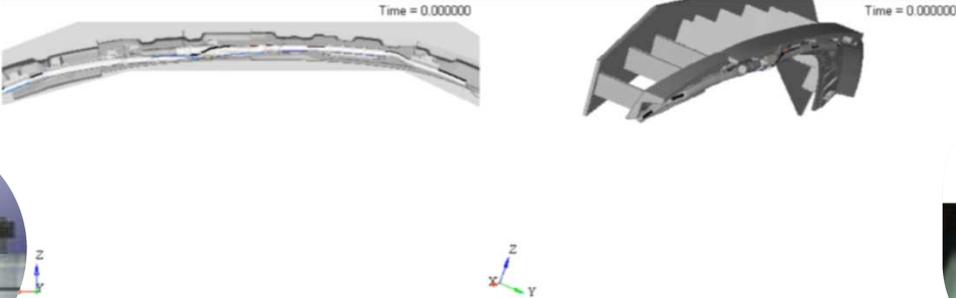
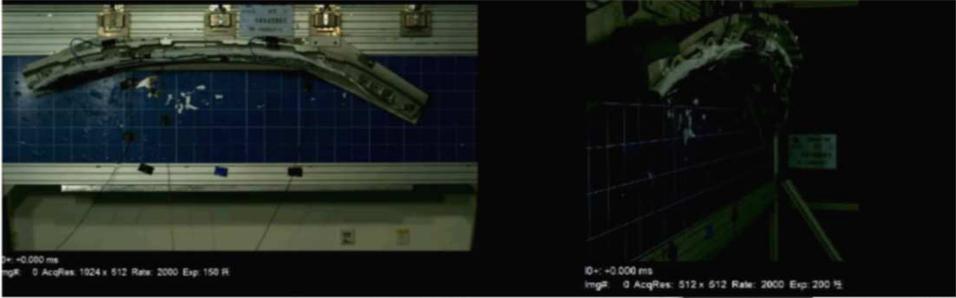
Airbag deployment IC

Static deployment



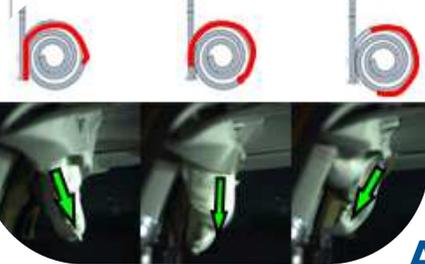
impact force from IC deploying

Acceleration when IC deploying



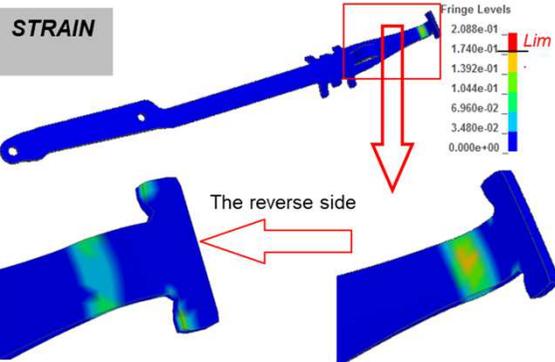
IC deployment with trim

IC folding influent deployment trajectory



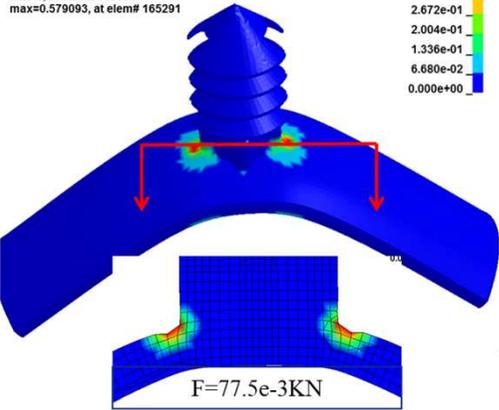
Airbag deployment IC

STRAIN



CLIP TENSILE STRESS ANALYSIS
Time = 29
Contours of Effective Plastic Strain
max IP. value
min=0, at elem# 148713
max=0.579093, at elem# 165291

Fringe Levels
4.008e-01
3.340e-01
2.672e-01
2.004e-01
1.336e-01
6.680e-02
0.000e+00

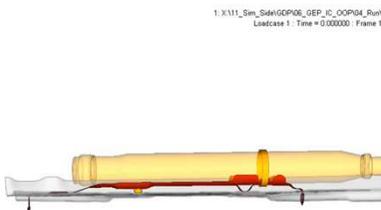
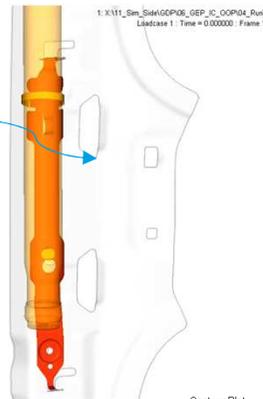
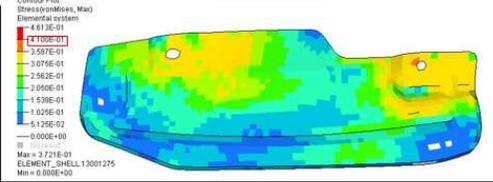


Grab Handle

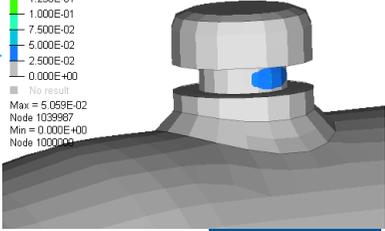
B-Pillar



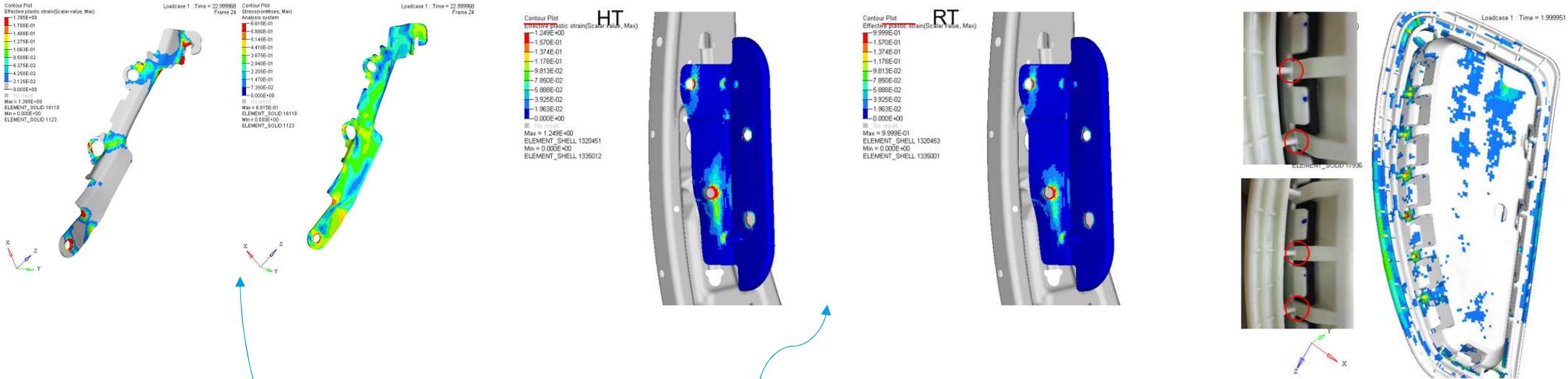
Max Von-mises stress



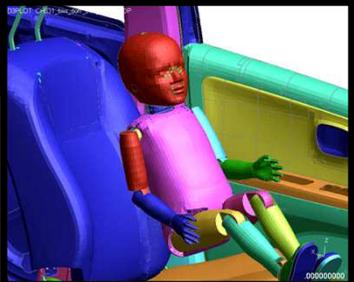
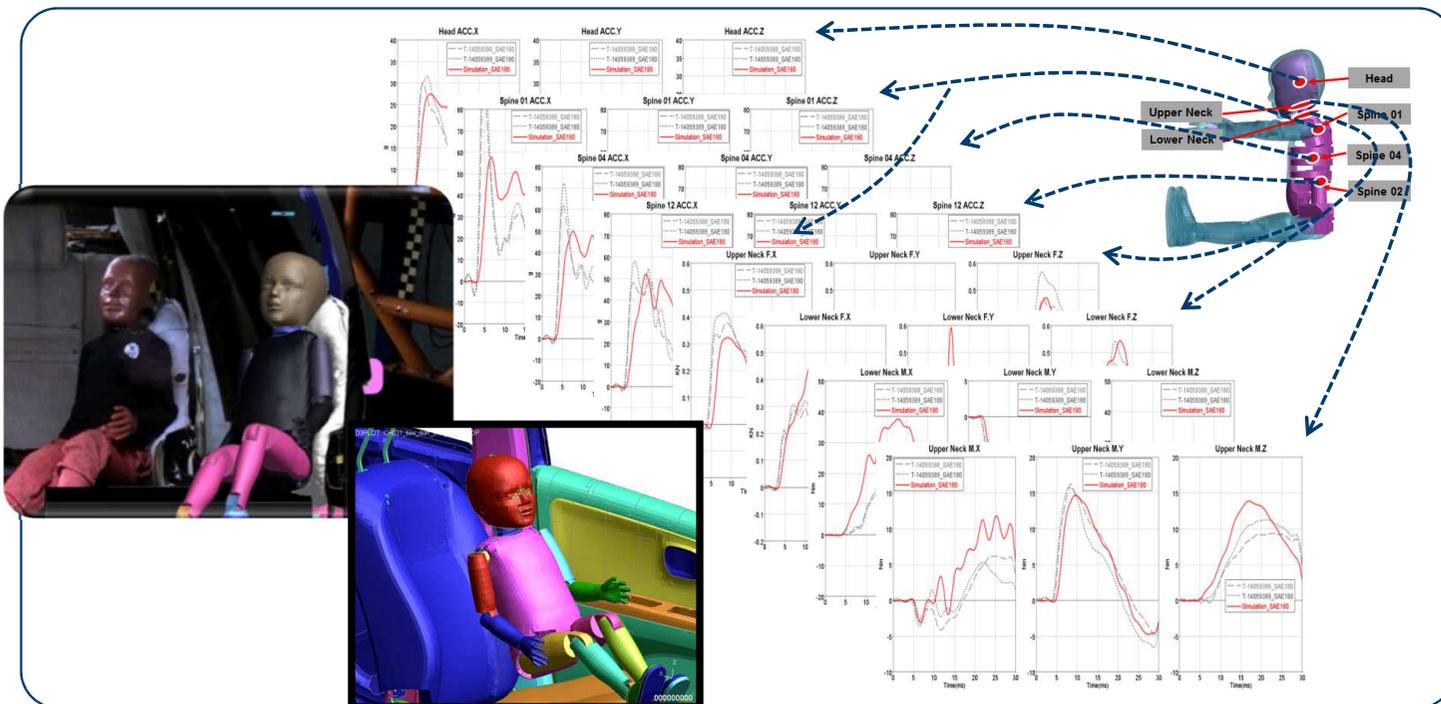
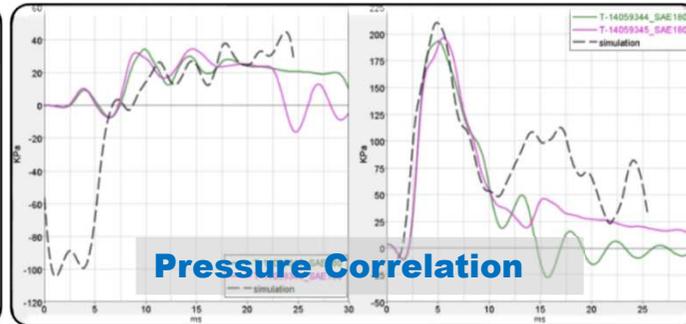
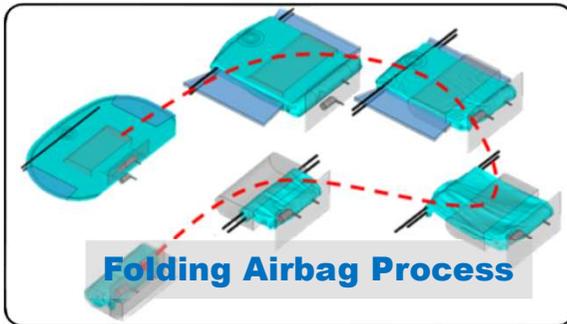
Contour Plot
Effective plastic strain (Scalar value, Max)
Simple Average
2.250E-01
2.000E-01
1.750E-01
1.500E-01
1.250E-01
1.000E-01
7.500E-02
5.000E-02
2.500E-02
0.000E+00



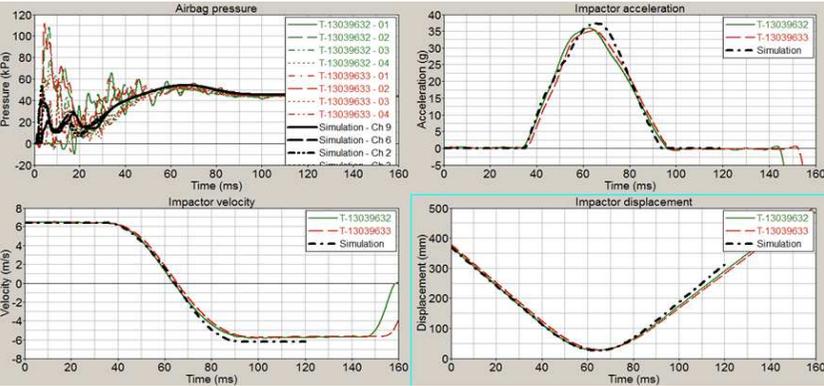
Airbag deployment SAB



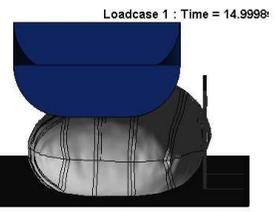
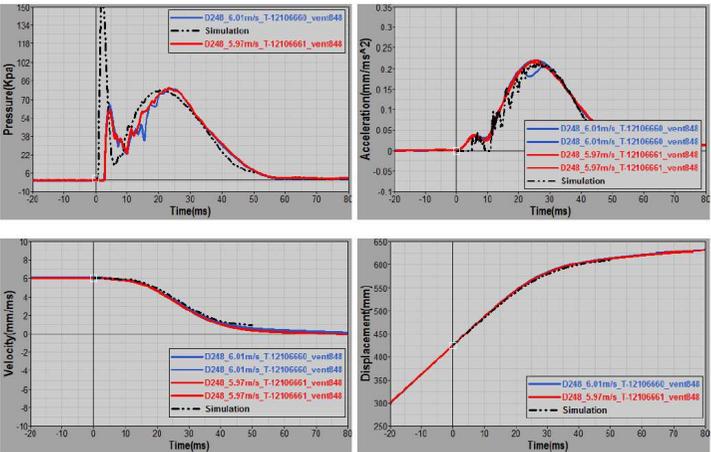
Airbag subsystem model OOP



Airbag subsystem model Validated for system



D3PLOT: LS-DYNA keyword deck by LS-PrePost



Application of LS-DYAN

Fatigue Analysis in Ls-Dyna

- Fatigue Simulation method
- Calibration Fatigue method
- Fatigue material properties
- Fatigue examples

Fatigue Analysis

What is fatigue?

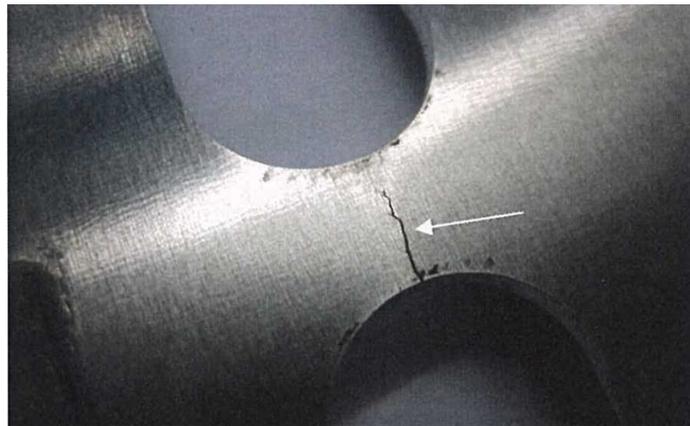
- ❑ Fatigue is a process in which damage accumulates due to the repetitive application of loads that may be well below the yield point.
- ❑ Fatigue is a complex process involving many steps but it can be broken down into initiation and propagation of fatigue cracks.
- ❑ It is estimated that fatigue failures are responsible for 90% of all metallic failures.
- ❑ For many years, fatigue has been a significant and challenging problem for engineers, especially for those who design structures such as aircrafts, railroad vehicles, automotives, bridges, pressure vessels, and cranes.



Fatigue Analysis

How to run fatigue analysis?

- ❑ Fatigue analysis can be performed in time domain and frequency domain.
- ❑ Two frequency domain approaches based on **random vibration theory and harmonic vibration (SSD) theory** have been implemented in LS-DYNA for fatigue and durability analysis.
- ❑ Recently we implemented time domain fatigue, including **one based on stress** and **the other based on strain** (further testing and validation needed)



Fatigue Analysis

S-N curve (high cycle, low stress)

*MAT_ADD_FATIGUE

Card 1	1	2	3	4	5	6	7	8
Variable	MID	LCID	LTYPE	A	B	STHRES	SNLIMT	SNTYPE
Type	I	I	I	F	F	F	I	I
Default	none	-1	0	0.0	0.0	none	0	0

- By *DEFINE_CURVE
- By equation

$$N \cdot S^m = a$$

$$\log(S) = a - b \cdot \log(N)$$

N: number of cycles for fatigue failure

S: stress

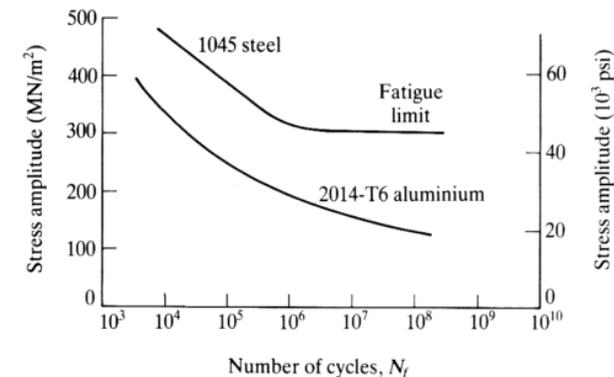
- Fatigue life of stress below fatigue threshold

SNLIMT *Fatigue life for stress lower than the lowest stress on S-N curve.*

EQ.0: use the life at the last point on S-N curve

EQ.1: extrapolation from the last two points on S-N curve

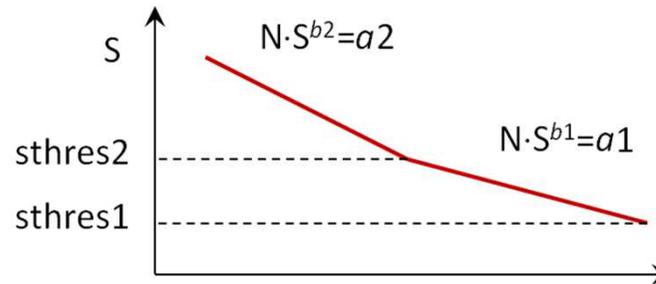
EQ.2: infinity.



Source of information: <http://www.efunda.com>

Fatigue Analysis

S-N curve (high cycle, low stress)



***MAT_ADD_FATIGUE**

Card 1	1	2	3	4	5	6	7	8
Variable	MID	LCID	LTYPE	A	B	STHRES	SNLIMT	SNTYPE
Type	I	I	I	F	F	F	I	I
Default	none		0	0.0	0.0	none	0	0

Card 2	1	2	3	4	5	6	7	8
Variable				Ai	Bi	STHRESi		
Type				F	F	F		
Default				0.0	0.0	none		

To define S-N curve with multiple slopes, the S-N curve can be split into multiple segments and each segment is defined by a set of parameters A_i , B_i and $STHRES_i$. Up to 8 sets of the parameters (A_i , B_i and $STHRES_i$) can be defined. The lower limit of the i -th segment is represented by the threshold stress $STHRES_i$.

Fatigue Analysis

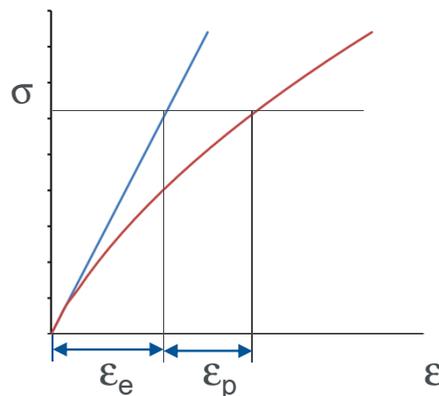
E-N curve (low cycle, high stress)

*MAT_ADD_FATIGUE_EN

Card 1	1	2	3	4	5	6	7	8
Variable	MID	KP	NP	SIGMAP	EPSP	B	C	
Type	I	F	F	F	F	F	F	
Default	none	none	none	none	none	none	none	

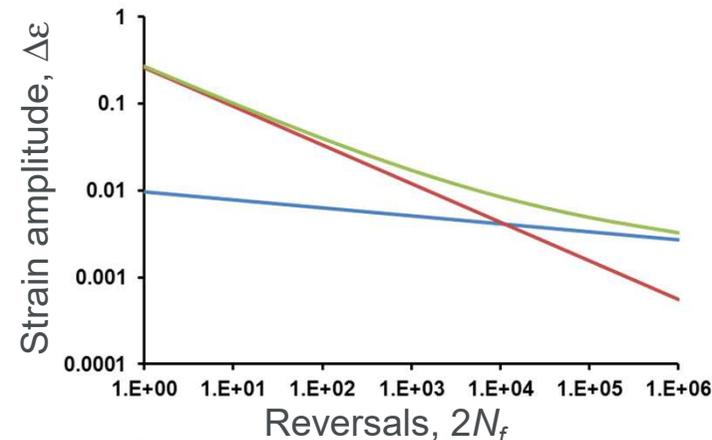
Cyclic stress strain curve

$$\epsilon = \frac{\sigma}{E} + \left(\frac{\sigma}{K'} \right)^{1/n'}$$



Local strain-life relationship

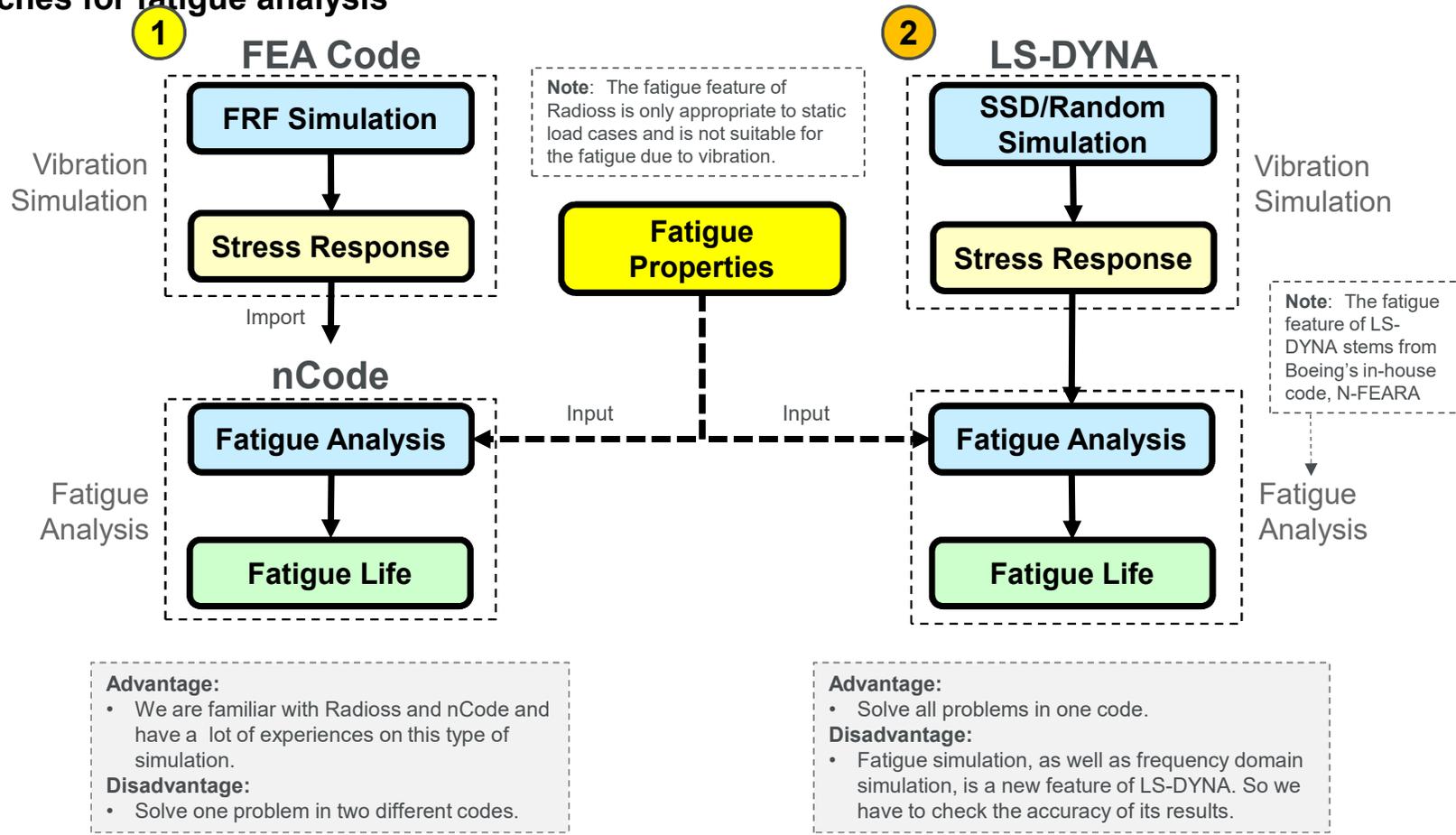
$$\frac{\Delta\epsilon}{2} = \frac{\sigma'_f}{E} (2N_f)^b + \epsilon'_f (2N_f)^c$$



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Fatigue Analysis

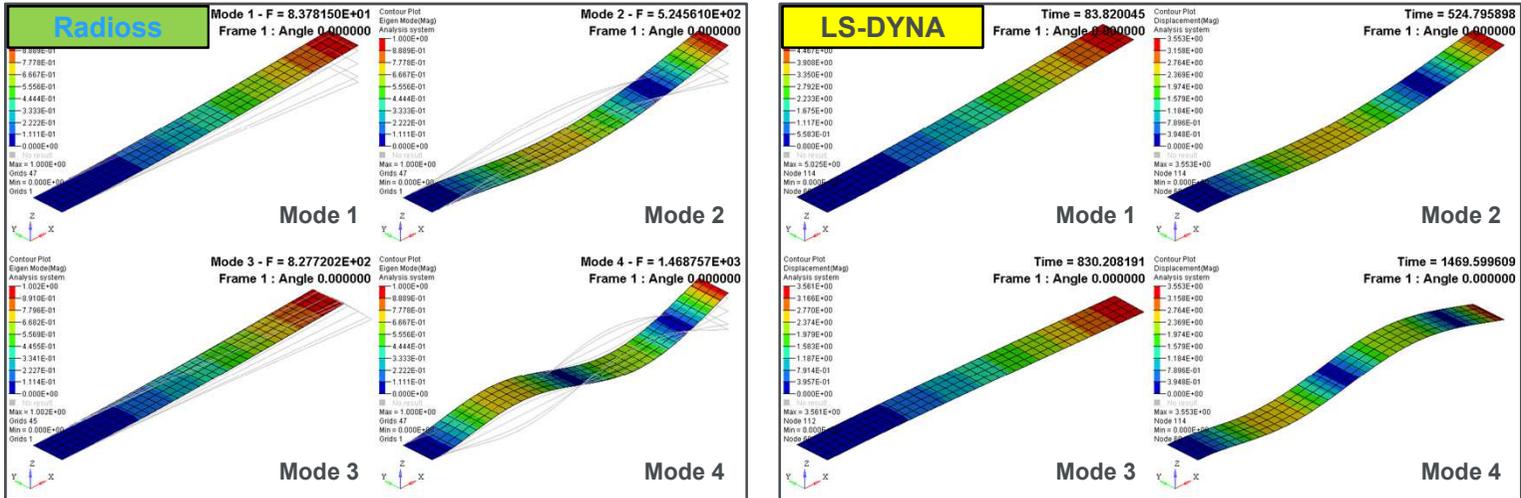
Two approaches for fatigue analysis



Fatigue Analysis

Comparison of Radioss (bulk data)/nCode and LS DYNA (implicit)

Natural Frequencies: simulation results



	Mode 1	Mode 2	Mode 3	Mode 4
Analytical Solution	83.28	521.93	832.78	1461.6
Radioss	83.78 100%	524.56 100%	827.72 100%	1468.76 100%
LS DYNA (ELFORM=2)	83.74 99.95%	524.54 99.99%	801.28 96.80%	1469.59 100.06%
LS DYNA(ELFORM=18)	83.82 100.05%	524.80 100.05%	830.21 100.30%	1469.60 100.06%
LS DYNA(ELFORM=20)	83.75 99.96%	524.27 99.94%	827.60 99.98%	1467.33 99.90%

Good Poor

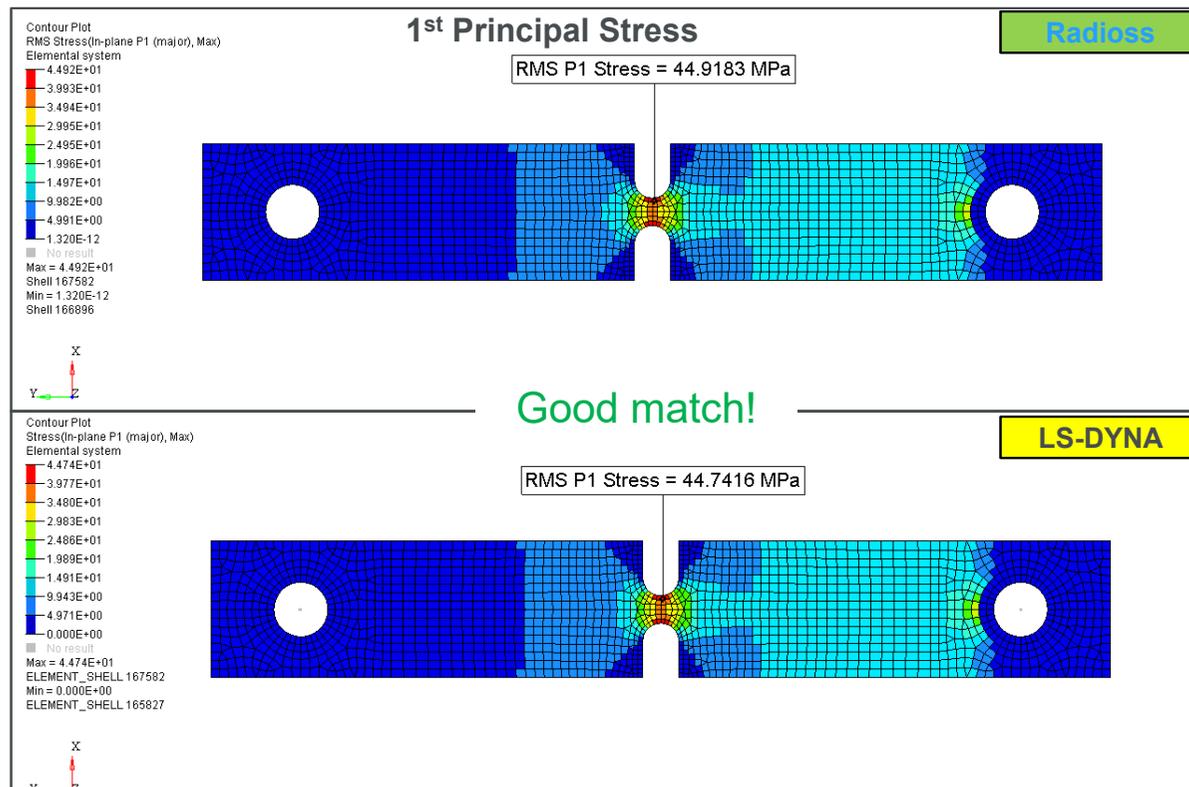
Note: ELFORM=2: Belytschko-Tsay (default)
 ELFORM=18: Fully Integrated linear DK quadrilateral or triangular shell
 ELFORM=20: Fully integrated linear enhanced shell → CQUAD4

Fatigue Analysis

Comparison of Radioss (bulk data)/nCode and LS DYNA (implicit)

Random Vibration: simulation results

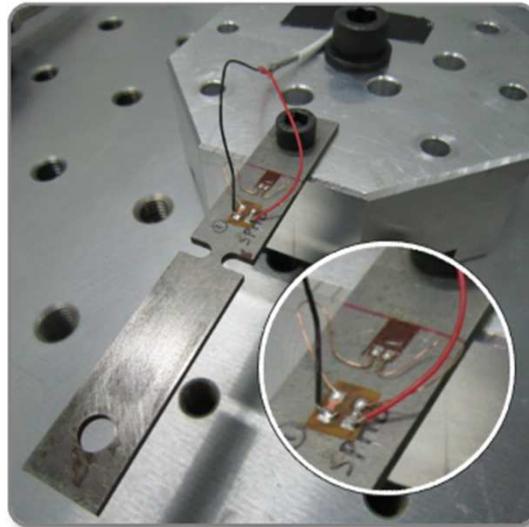
- The Root Mean Square (RMS) stress of the specimen is calculated by *Radioss* and *LS-DYNA*, respectively.



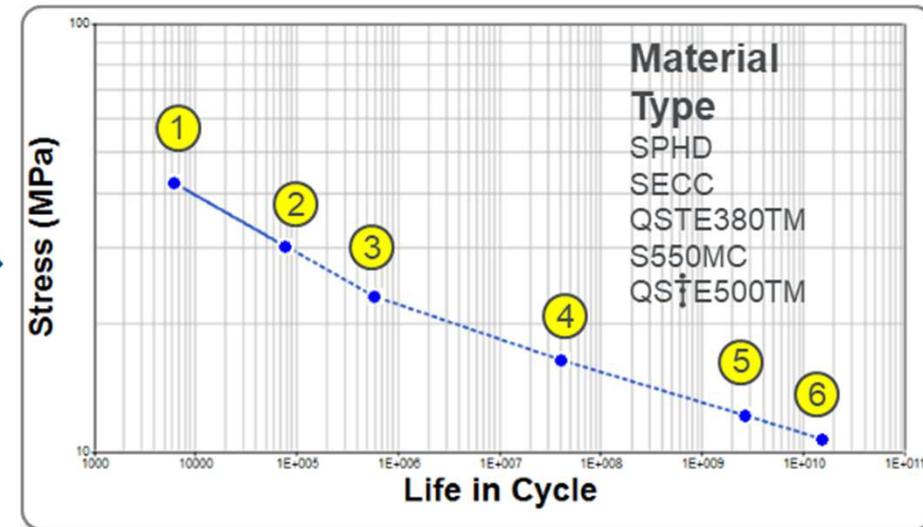
Fatigue Analysis

Measure the fatigue material property

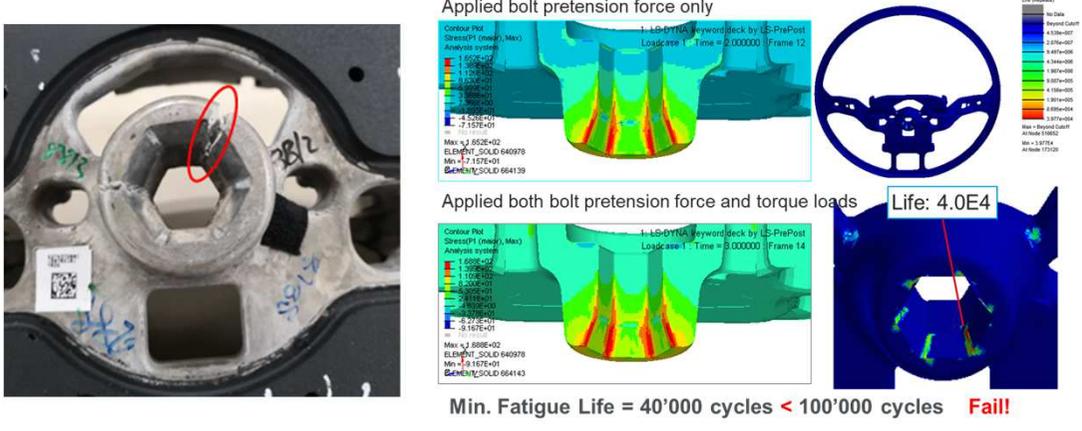
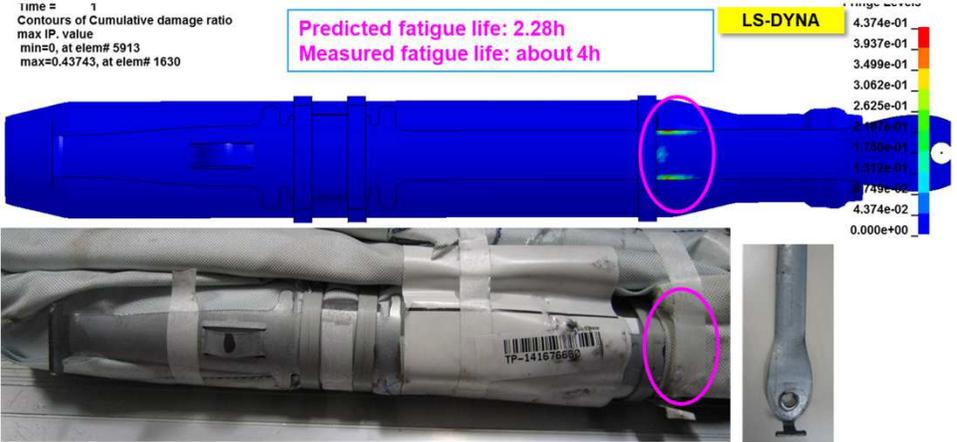
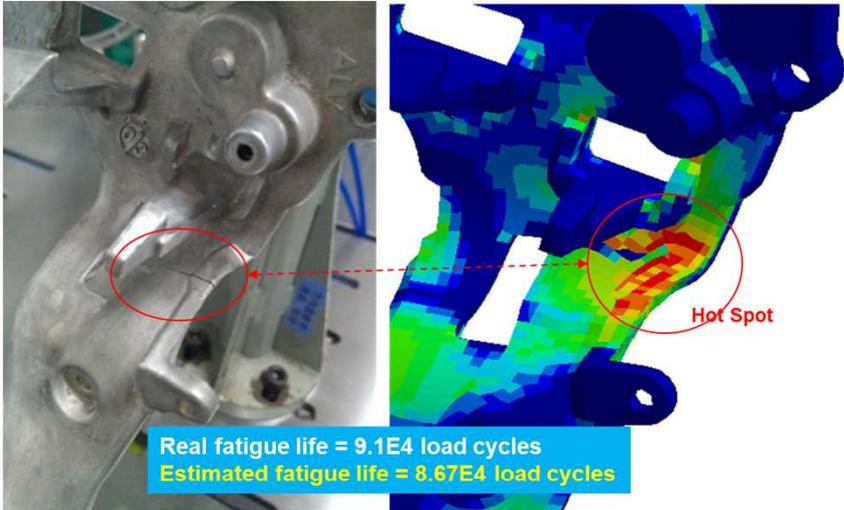
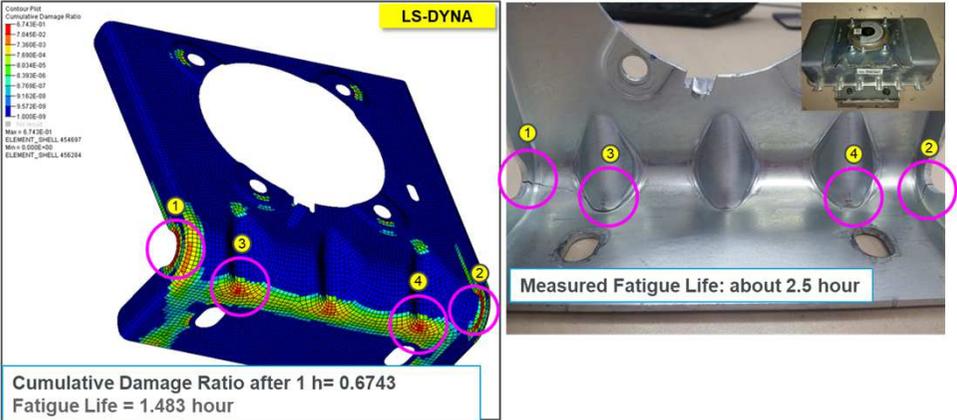
Shaker-based fatigue test



Fatigue material property data base



Fatigue Analysis



Application of LS-DYAN

Acoustic Simulation in Ls-Dyna

- Introduce
- Application
- Steering wheel

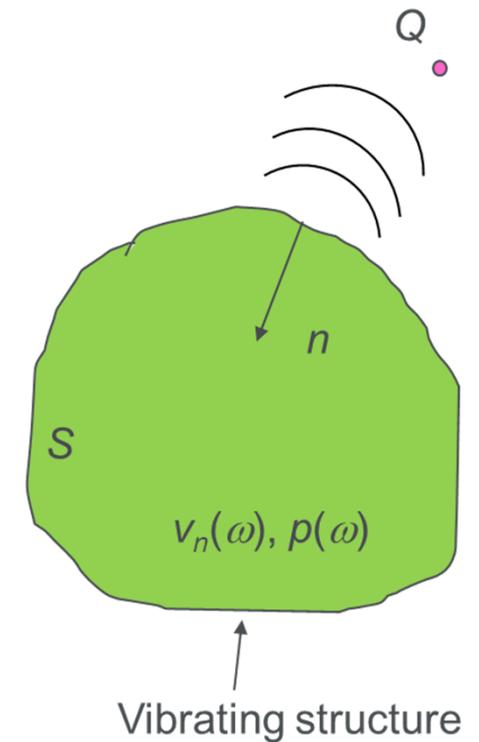
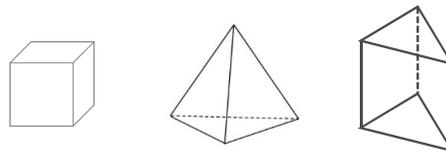
Acoustic Simulation introduce

Time domain acoustic solver in LS-DYNA

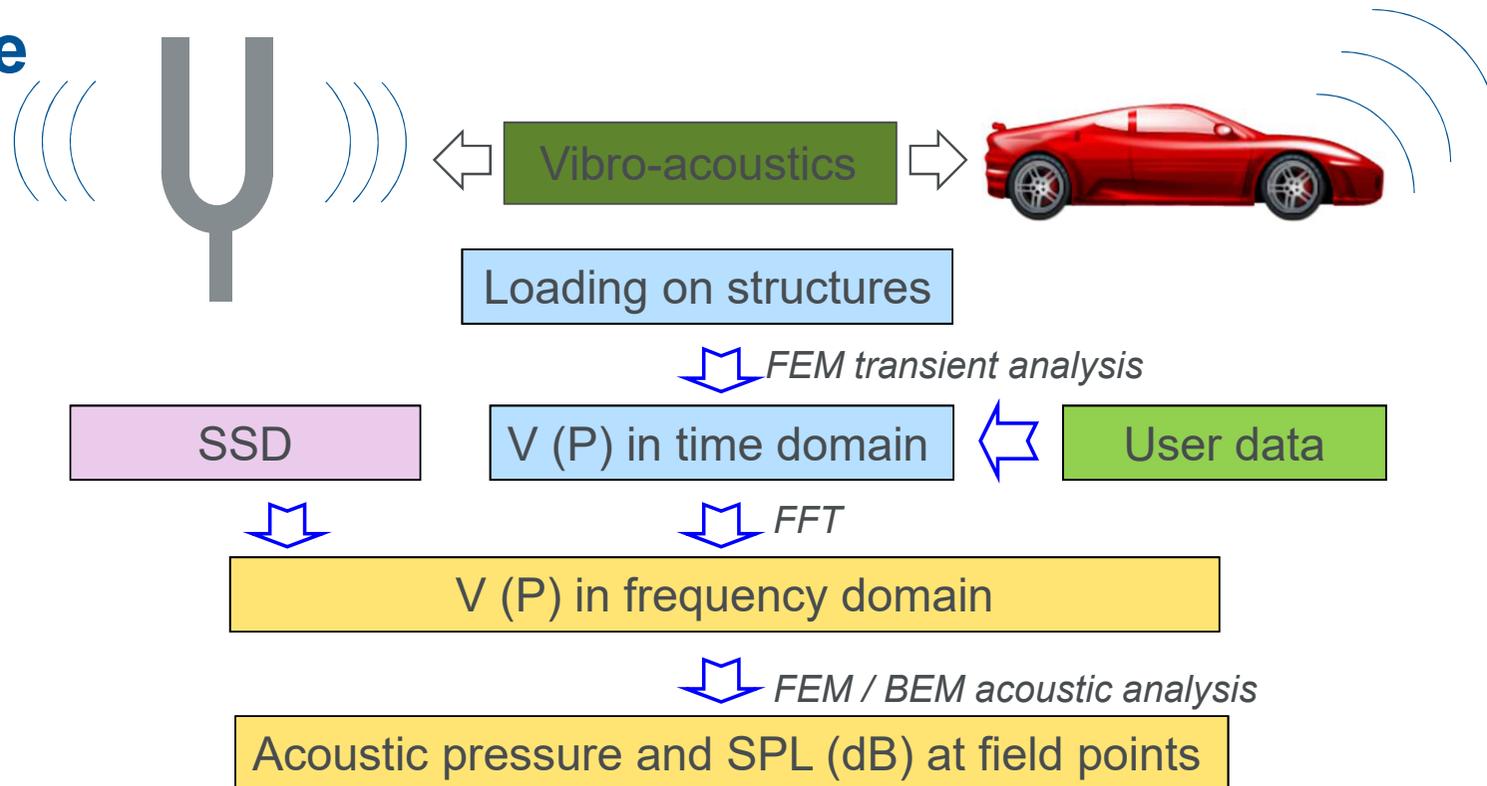
- ✓ MAT_ACOUSTIC with solid element 8 or 14

Frequency domain acoustic solver in LS-DYNA

- ✓ FREQUENCY_DOMAIN_ACOUSTIC_BEM
 - Rayleigh method
 - Kirchhoff method
 - Collocation BEM
 - Variational indirect BEM
- ✓ FREQUENCY_DOMAIN_ACOUSTIC_FEM
 - Hexahedron
 - Tetrahedron
 - Pentahedron
- ✓ SEA (*ongoing development*)



Acoustic Simulation introduce



National Taipei University of Technology, Taiwan: Guo-Ding Huang, Hsiu-Ying Hwang, Xijun Wang, “*Vibration Testing and Analysis for a Midsize Electric Bus*”, Proceedings of the 19th National Conference on Vehicle Engineering, Nov. 14, 2014, TIIT, Jhongli, Taiwan.

Acoustic Simulation introduce

BEM (accurate)

- Indirect variational boundary element method
- Collocation boundary element method
 - Burton-Miller formulation
 - Sound power and radiation efficiency are computed

They used to be time consuming

A fast solver based on domain decomposition

MPP version

Approximate (simplified) methods

- Rayleigh method
- Kirchhoff method

Assumptions and simplification in formulation

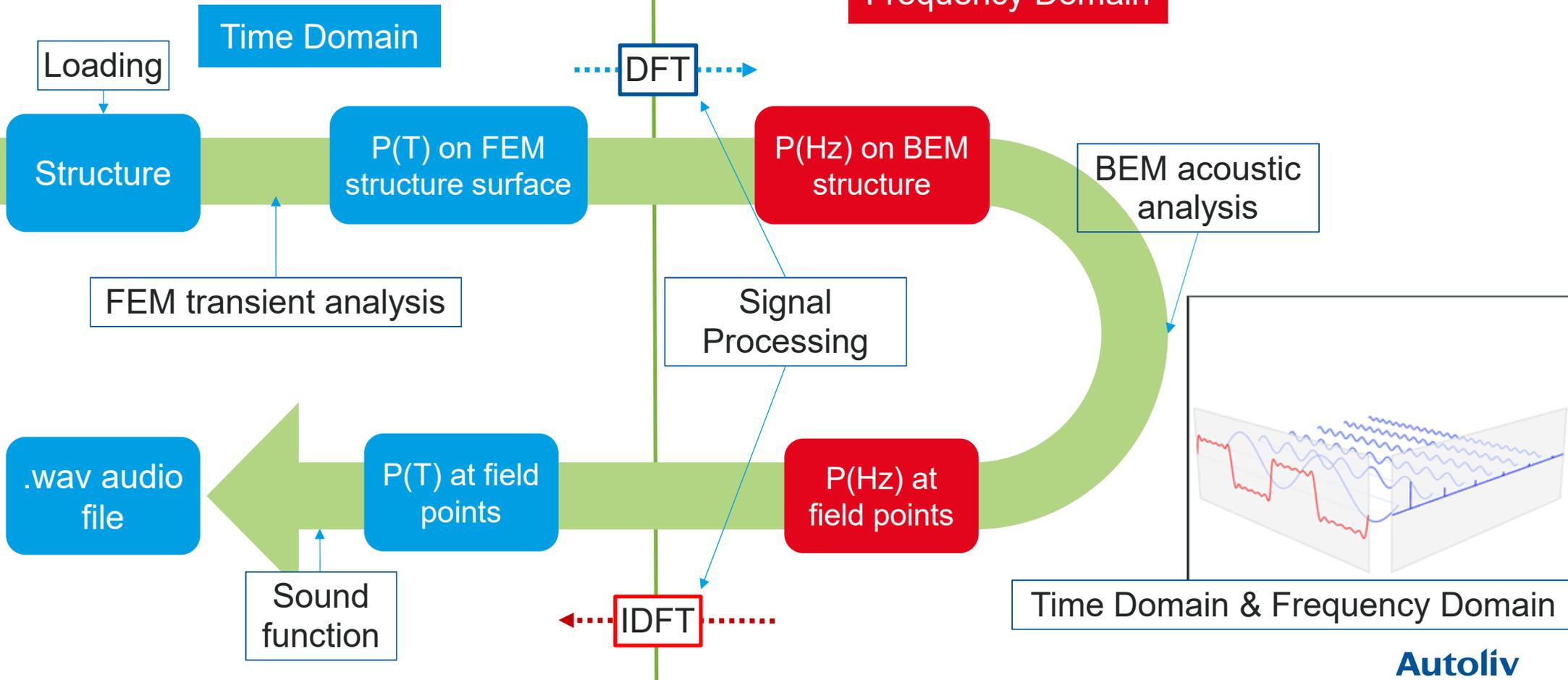
Very fast since no equation system to solve

Acoustic Simulation introduce

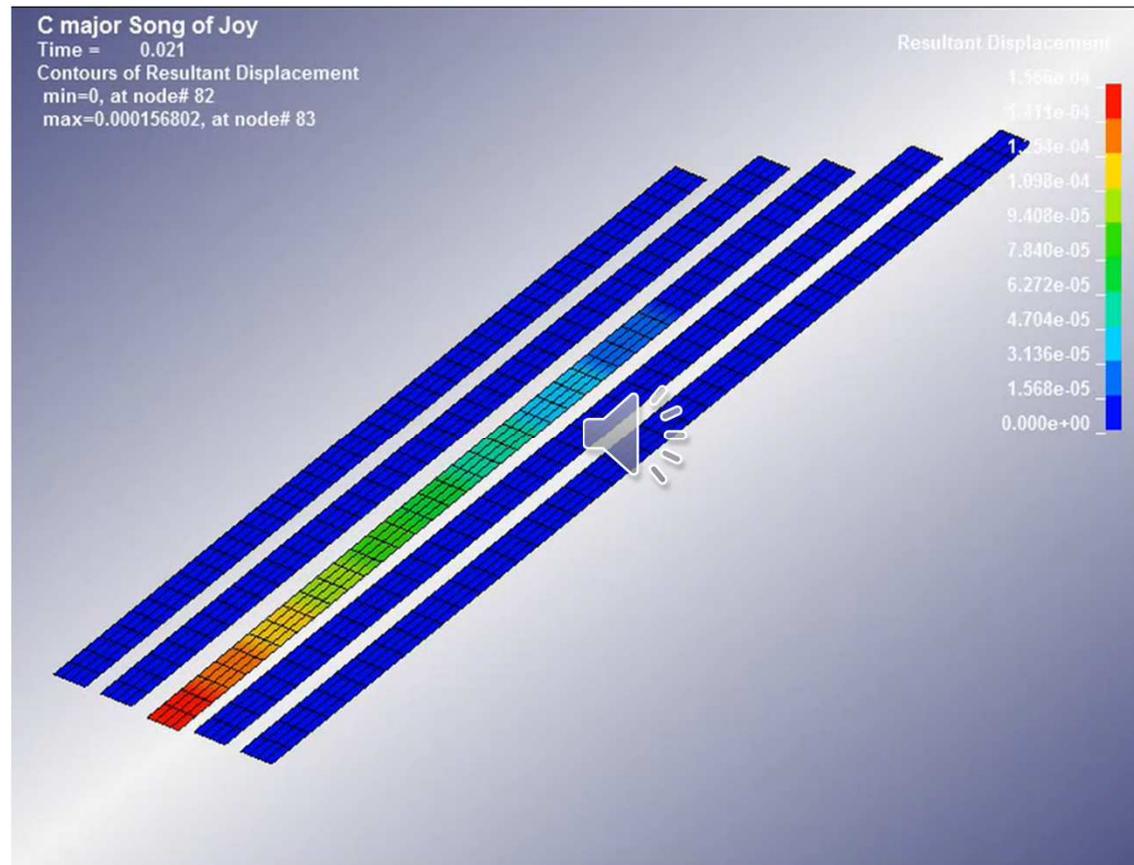
Technical Route

Frequency Domain

Time Domain



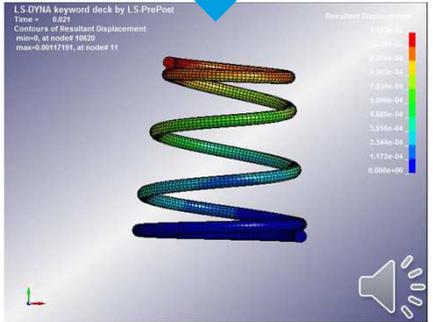
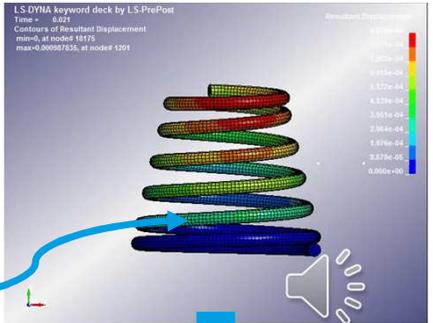
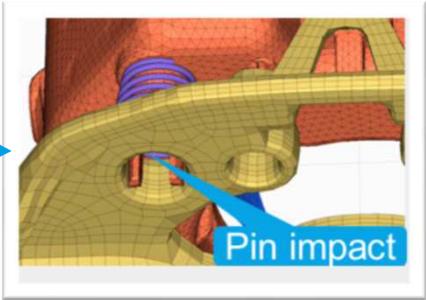
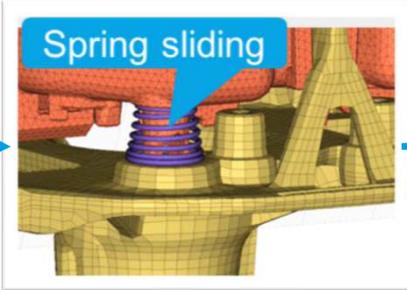
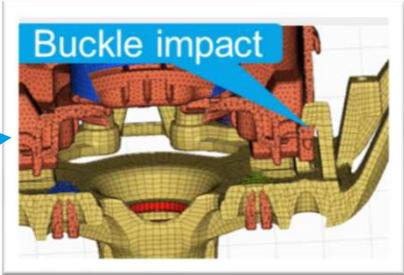
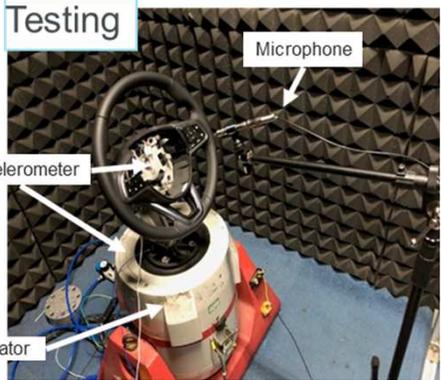
Acoustic Simulation Music Box (minimalism version)



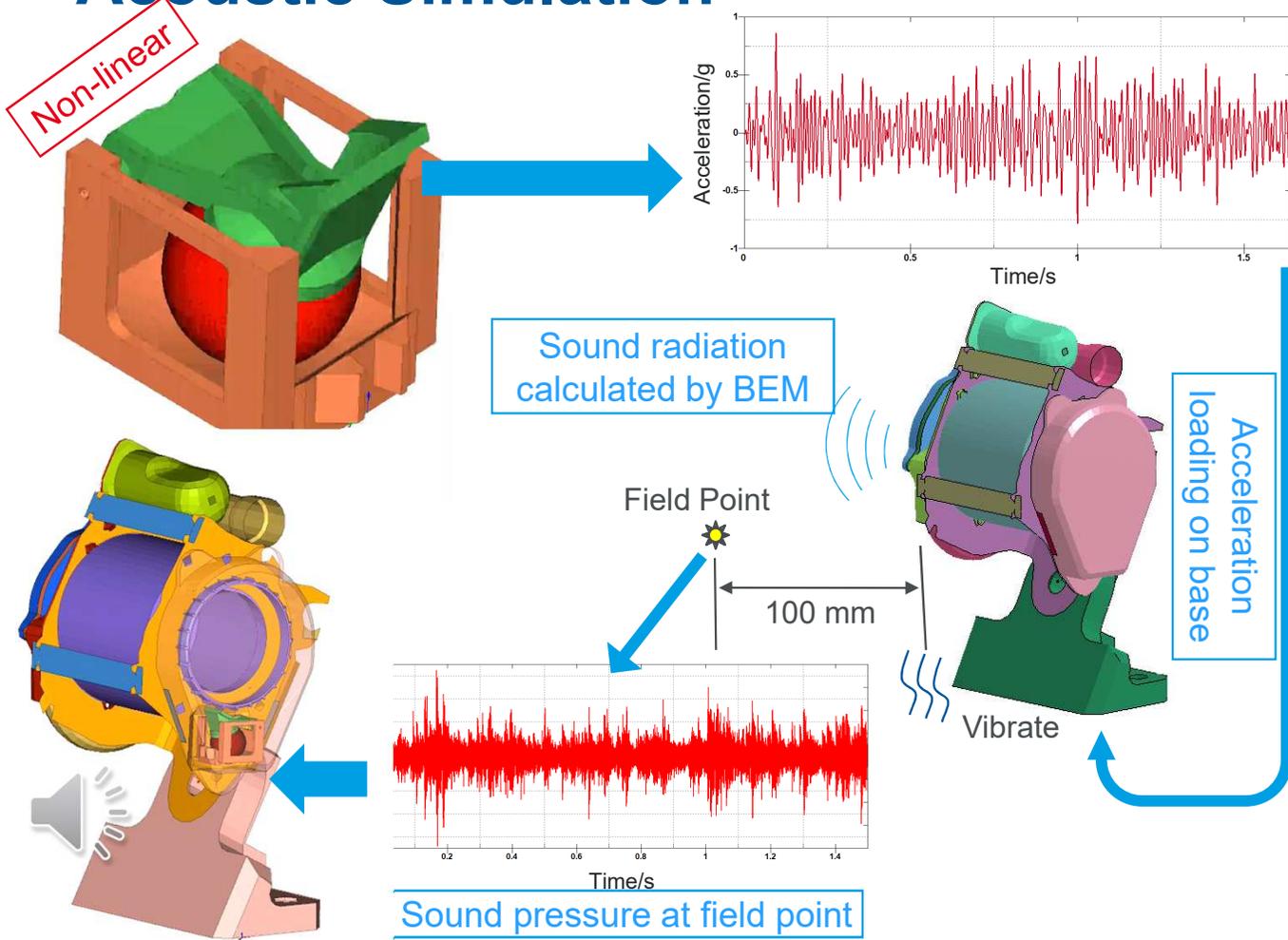
Acoustic Simulation

Dynamic analysis for noise source identification

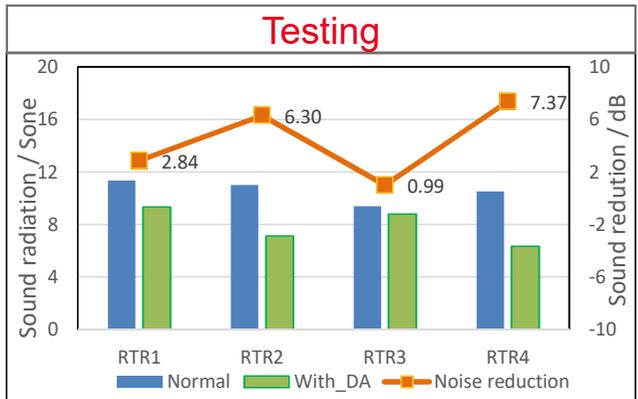
- Transient analysis for sound simulation



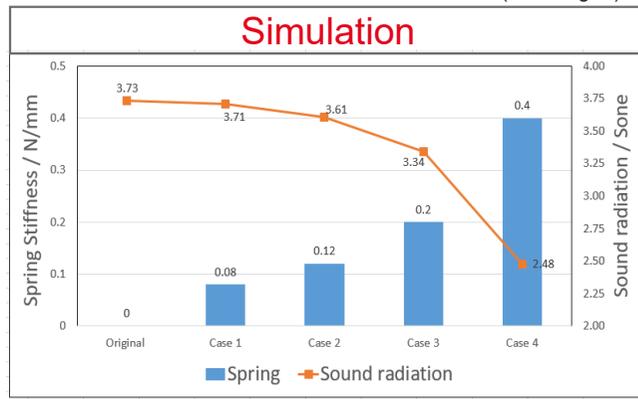
Acoustic Simulation



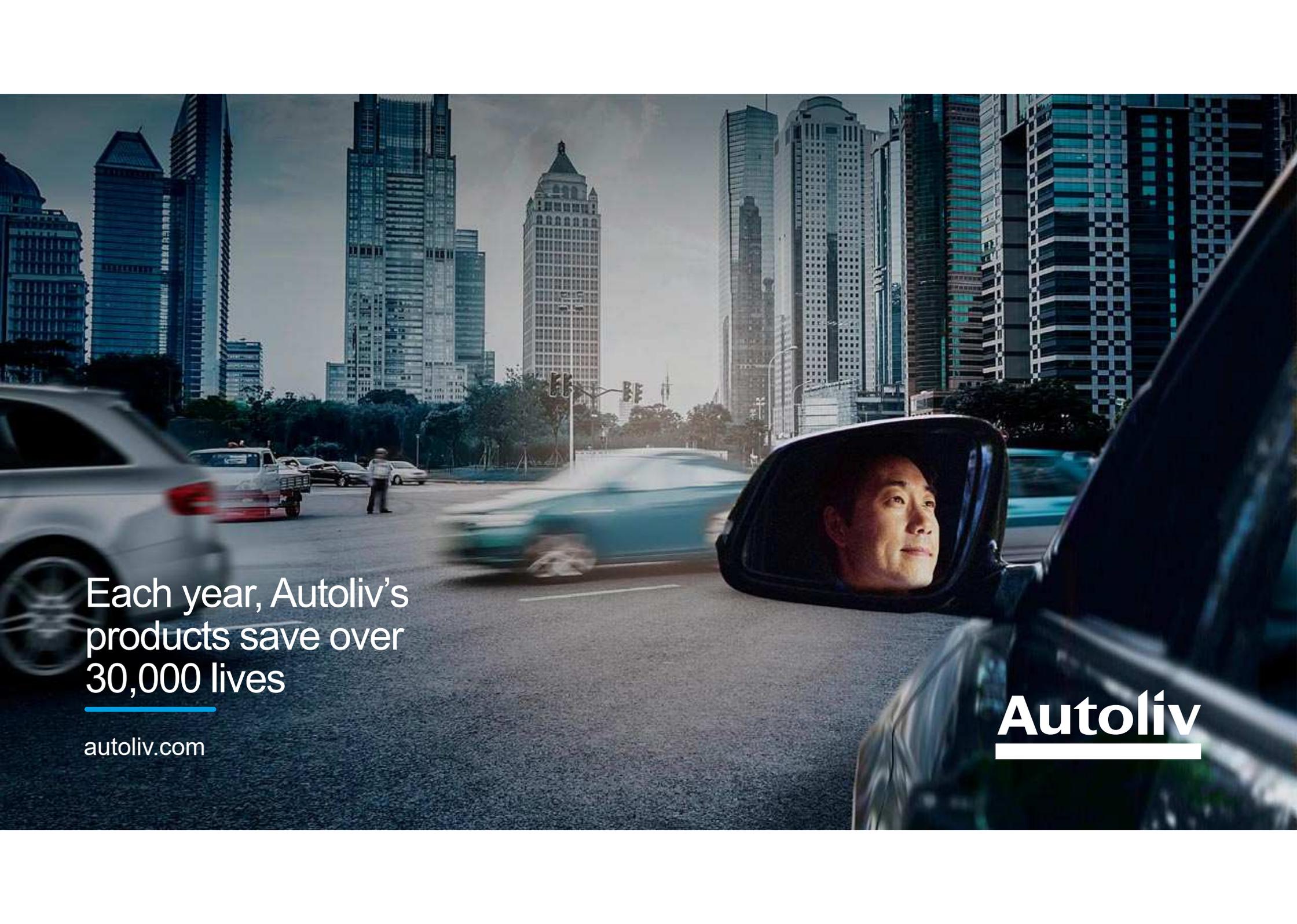
CS Rattle noise is caused by highly non-linear, can only be simulated by transient dynamic analysis.



Noise reduction: 4.37 dB (4 averaged)



Noise reduction: 5.89 dB



Each year, Autoliv's
products save over
30,000 lives

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